



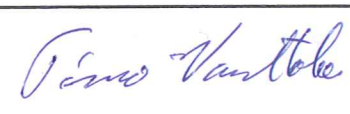


Adjusting the valve boundary condition in Olkiluoto 1 load rejection test calculated using TRAB-3D

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| Authors | Malla Seppälä |
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| <p>Summary</p> <p>At VTT, one of the programs used for transient calculations is 3D coupled neutronics/thermal hydraulics code TRAB-3D. It is validated against international benchmarks and measured data from a load rejection test performed on Olkiluoto 1 nuclear reactor in 1998. The original calculation on the load rejection test was based on measurement data with 0.2 s sample interval. In 2001, new data with 0.02 sample interval was received from TVO.</p> <p>The objective of this study was to determine whether the accuracy of the load rejection test calculation could be improved by imposing new valve boundary conditions in accordance with the measurement data received in 2001. The boundary condition was varied and results compared to the original calculations and measured data.</p> <p>It was discovered that the importance of the new measurement data is minor. Changes on the boundary condition needed to be substantial in order to increase the accuracy of the calculation and even then the impact could be seen only for a couple of seconds at the beginning of the calculation.</p> | |
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|  Malla Seppälä Trainee Research Scientist |  Antti Daavittila Team Leader |
|  Timo Vanttola Technology manager | |
| VTT's contact address P.O. Box 1000, FI-02044 VTT, Finland | |
| Distribution (customer and VTT) Fortum: M. Antila, R. Teräsvirta TVO: M. Solala, K. Ranta-Puska STUK: R. Sairanen, R. Mattila TKK: P. Aarnio LUT: R. Kyrki-Rajamäki VTT: A. Daavittila, T. Vanttola, A. Hämäläinen, H. Rätty, E. Syrjälähti, M. Anttila, E-K. Puska, V. Suolanen, S. Tähtinen | |
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1 Introduction

During the designing process of a nuclear power plant many kinds of safety calculations are made to ensure the durability of the plant and to discover any defects left unnoticed. One category of calculations is transient calculations in which severe transient scenarios are studied in order to determine whether the response of the reactor is in accordance with regulations. At the Technical Research Center of Finland (VTT) three dimensional transient calculations are performed using two programs, TRAB-3D and HEXTRAN, depending on the geometry of the fuel assembly. Both programs have been developed at VTT. An important part of developing a transient code that models nuclear reactors is the validation of the program using international benchmarks or data from experiments carried out on domestic nuclear reactors.

The objective of this study is to examine one of the validation calculations performed on TRAB-3D, a load rejection test performed on the Olkiluoto 1 reactor in Finland in 1998, and to determine whether the calculation could be enhanced using new, more detailed measurement data received from TVO in 2001. The original calculation was based on data with 0.2 s sample interval. In the new data the sample interval is 0.02 s. The focus of the new calculation is on a valve opening-closing mismatch that occurs during the first few seconds of the test initiating a disturbance in the system pressure. The mismatch is modeled in TRAB-3D with a boundary condition in the input. First the exact mismatch values from the data are put to use in the boundary condition and the results are assessed in respect to the original calculation to evaluate the importance of the new data. Second the mismatch is adjusted to maximize the accuracy of calculated pressure in respect of measured pressure during the first second of the calculation. Last the accuracy of calculated pressure is maximized during first three seconds of the calculation. The latter two calculations are performed to determine the sensibility of the calculation regarding the mismatch.

2 Models and solving methods in TRAB-3D

In a nuclear reactor core the coupling of neutronics, heat transfer and thermal hydraulics is very strongly present. It is therefore of vital importance to describe this phenomenon in the dynamic codes in order to gain a feasible model of the core. At VTT coupled three dimensional transient codes have been developed, validated and used since the mid eighties.

TRAB-3D [1, 2] is a stand-alone BWR dynamics code, whose core model is based on the 3D hexagonal core model in HEXTRAN [3], and circuit and system models are adopted from one-dimensional code TRAB [4]. TRAB-3D can be applied to transient and accident analyses of boiling (BWR) and pressurized water (PWR) reactors with rectangular fuel bundle geometry. The code has been validated against international benchmarks and actual measured data from real plant transients and it is in active use at VTT.

2.1 Neutronics

In TRAB-3D neutronics are modeled with two-group diffusion equations which are solved by nodal expansion method in x-y-z-geometry. The nodal expansion is based on the steady-state solution method for hexagonal core model in the HEXBU-3D simulator. The basic principle of the solving method is the decoupling of the diffusion equations, which is accomplished by separating the equations using two spatial modes. The characteristic solutions of the spatial modes construct the two group fluxes. There are two types of characteristic solutions, asymptotic and transient. The fundamental, asymptotic mode is a well-behaving function within a node and it can be approximated using polynomial functions, whereas the transient mode has a large buckling in LWR cores and therefore needs to be approximated by exponential functions. The interaction of adjacent nodes is handled through continuity conditions at the interfaces. The dynamic equations include six groups of delayed neutrons. [5]

A two-level iteration scheme is used to solve the nodal equations. In the inner iteration only one unknown, the average fundamental mode, is determined. The

nodal flux shapes are enhanced in the outer iteration by recalculating the coupling coefficients. Cross-sections are calculated using polynomial fittings to coolant temperature, density and soluble boron density. Homogenized cross sections are created with the CASMO-4 cell burnup code. [1]

2.2 Thermal Hydraulics

The thermal hydraulics model and its solution method are adopted from the one-dimensional transient code TRAB. Flow channels are parallel and they are connected freely to one or several fuel assemblies. The principal equations represent the conservation of the masses of water and steam, respectively, total enthalpy and total momentum. Additional equations for disequilibrium of evaporation and condensation, slip and one- and two-phase friction can also be utilized. Mass distribution through flow channels is determined from the pressure balance in the core. The phase velocities are determined using slip ratio or the drift-flux formalism. Properties of water and steam are calculated locally as rational functions of pressure and enthalpy.

During the thermal hydraulic iteration one-dimensional heat transfer is calculated for an average fuel rod in each fuel assembly. Fuel rod and cladding are discretised with several radial mesh points and the calculations are performed at equidistant axial elevations. Heat conduction is solved according to Fourier's law with temperature dependent thermal properties of fuel pellet, gas gap and fuel cladding and with different heat transfer coefficients for different hydraulic regimes.

TRAB-3D includes a 1D model for BWR's thermal hydraulics inside the pressure vessel, as well as models for pumps, steam lines, and control systems. The circuit components of TRAB-3D are shown in Figure 1. When TRAB-3D is used for PWR transient calculations, it is coupled with a 1D thermal hydraulic system code SMABRE for the hydraulic circuit. [1]

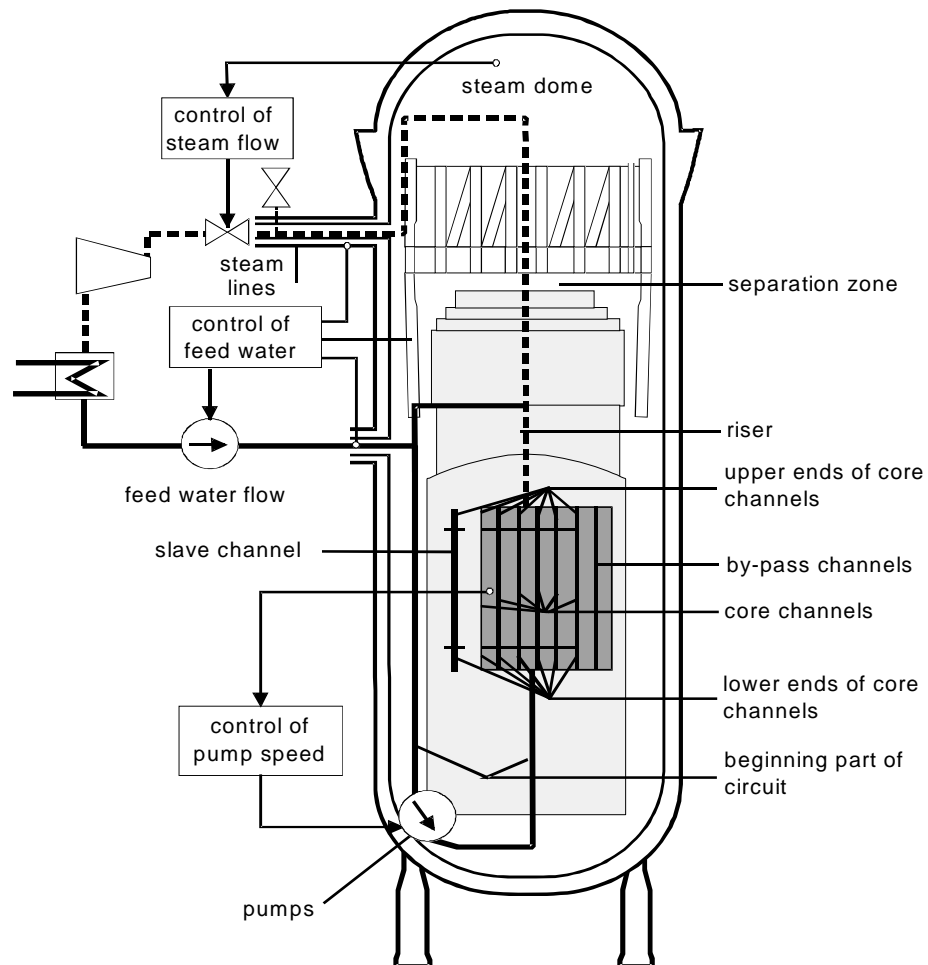


Figure 1. TRAB-3D BWR pressure vessel model. [1]

3 Load rejection test

3.1 Test description

The load rejection test was performed on Olkiluoto1-reactor on June 16th 1998 during the start-up testing after its modernization and power uprating. The purpose of the test was to show that the plant can endure a shift from full 2500MW operation power to a 30% power level in case of external load rejection. In this situation the plant is supposed to feed an in-house load and dump excess steam to the condensers. These actions were carried out through a partial scram, in which one scram rod group was inserted hydraulically in a few seconds and another slowly electrically taking 260 seconds, and by slowing the main circulation pump to minimum speed. The plant functioned as expected. [1]

The test offers excellent material for testing of 3D codes, because the transient is asymmetric in the core and there are measurements from local power range monitors available on several axial and radial locations in the core as well as measurement values of bundle flows for several bundles.

At the initial stage of the test the plant was on 2500 MW operation power, the turbine valves were open and the dump valves closed. At the beginning turbine valves close and dump valves open. At the same time a partial scram takes place, as one rod group is inserted hydraulically and another slowly. Feed water pumps operate both under automatic control and under manual control by the operators. The plant stabilized on a 30% house turbine operation power level. [6]

3.2 Calculation using TRAB-3D

Several modifications were initially done on TRAB-3D in order to calculate the load rejection test. The first full core model on TRAB-3D was constructed, since the partial scram was not half-core symmetric. The thermal hydraulic model was altered to take into account the impact of the partial length Atrium fuel rods on the flow geometry. A simple detector model was coded approximating local power range monitors (LPRM) with averaging the thermal flux of the four nodes adjacent to the detector. Average power range monitor (APRM) values are calculated from 28 LPRM values. As the main object of the calculation was to evaluate the capability of TRAB-3D to model 3D core phenomena, no modifications were made on coolant circuit model or controller models. The values of the feed water flow were set as transient boundary conditions because of the manual operator actions during the test and due to the absence of feed water controller in the present TRAB-3D input for TVO reactors. Feed water enthalpy and pump speed were also used as boundary conditions. [7]

In the initial test measurement data the closing of the turbine valves and the opening of the dump valves appeared simultaneous. After some test calculations, a 0.05 s mismatch in the closing and opening of the valves was chosen due to the relatively long sample interval 0.2 s, and the total time taken in opening and closing the valves was fixed to 0.15 s. These values were also set as boundary conditions for the calculation. The load rejection test was calculated for 400

seconds with TRAB-3D. Agreement with measured 3D time-dependent LPRM data was good. [1, 8]

4 New calculations

In 2001 new, more specific measurement data of the load rejection with a 0.02 s sample interval was received. The objective of this study is to determine whether the accuracy of the calculation of the load rejection test could be improved by imposing more detailed boundary conditions for the turbine and dump valves in the beginning of the calculation. In addition, the sensitivity of the calculation in respect of the opening and closing of valves was evaluated and thereby the significance of the new data.

In the current TRAB-3D input opening and closing of valves is modeled by one variable, which represents the fraction of openness of all the turbine and dump valves. This is because in the present calculation the four steam lines are lumped as one. Therefore, if no mismatch was related to the closing of the turbine valves and the opening of the dump valves, the value of the valve variable, A2VAL1, would stay one through the first few seconds of the calculation. In the initial calculations the mismatch was modeled by a plateau between the beginning of the closing of the turbine valves and the end of the opening of the dump valves, shown in Figure 3. The constant value of the valve variable on the plateau was determined by varying it and comparing the calculated pressure to measured system pressure.

For the new calculation, the fraction of openness of the valves in the measurement data was plotted to determine more detailed boundary conditions, see Figure 2. In the new, exact boundary conditions mismatch is modeled the same way as in the original calculation, by a plateau. The value of the valve variable on the plateau was determined by comparing calculated and measured pressures.

In order to evaluate the sensitivity of the calculation and to find out the optimal mismatch values, the calculation was performed with varying valve boundary

condition values. The mismatch and time taken in opening and closing the valves was varied freely, without paying any attention to the values gained from measurement data. The accuracy of the calculated system pressure was chosen as criterion for the quality of the boundary condition. At first only the first second of the calculation was considered when comparing the pressures and later first three seconds were taken into account.

Closer inspection of the new data revealed that the starting point of the calculation needed to be readjusted in respect to the measured data to allow comparison. The zero-time of the calculation was set to the beginning of the valve opening/closing and the calculation started 0.1 seconds before it. In the initial calculation, the zero-time of the calculation corresponded to 58.2 s in the measured data. It was corrected to 58.28 s in accordance with the new data.

5 Results

The valve boundary condition is modelled in tabulated form in the input. The values of the variable, A2VAL1, used in the calculations are shown in Table 1 and they are graphed in Figure 3.

Table 1. Valve boundary condition values.

| | | | | | |
|-------------------------------|----------|------|--------|--------|------|
| Original calculation | Time (s) | 0.1 | 0.15 | 0.25 | 0.3 |
| | A2VAL1 | 1.0 | 0.6667 | 0.6667 | 1.0 |
| Exact boundary conditions | Time (s) | 0.1 | 0.11 | 0.13 | 0.14 |
| | A2VAL1 | 1.0 | 0.5 | 0.5 | 1.0 |
| 1s-fitted boundary conditions | Time (s) | 0.21 | 0.34 | 0.4 | 0.57 |
| | A2VAL1 | 1.0 | 0.74 | 0.74 | 1.0 |
| 3s-fitted boundary conditions | Time (s) | 0.2 | 0.3 | 0.8 | 1.1 |
| | A2VAL1 | 1.0 | 0.87 | 0.87 | 1.0 |

Figures of the most interesting variables, pressure, main circulation flow, steam flow out of steam line and feed water flow are presented in Figures 4-7 for the first ten seconds of the calculation for the different boundary conditions. Measured data and data from original calculation are also graphed in the same figures to allow comparison. In addition, results from a local power range monitors (LPRM16) located near a hydraulic scram rod group at four heights axially are shown in Figure 9 and results from an average power range monitor (APRM1) based on 28 LPRM measurements in Figure 8. Relative LPRM values are scaled with an arbitrary initial value to allow visualization of four axial values in the same figure.

Examining system pressure in Figure 4 shows that the exact mismatch determined from measured data is too rapid to create the rise in pressure observed in the measured pressure in the beginning of the test. During the first four seconds, the calculation performed using the exact boundary condition is less accurate than the original calculations. For the calculation in which pressure was optimized for the first second of the calculation, the calculated pressure is, as expected, quite consistent with measured pressure for the first second and therefore more accurate than the pressure in the original calculation. In the 3s optimized calculation calculated pressure follows the measured pressure well for almost three seconds. However, after four seconds all the calculated pressures converge and no deviation can be seen between the original calculation and new calculations.

Steam flow out of steam line in Figure 5 shows differences between the calculations for the first three seconds of the transient but little improvement can be seen in its accuracy. In the other variables, the new calculations show little variation from the original calculation.

A longer 300 s calculation was performed using the last valve boundary condition, in which pressure was optimized for three seconds. Figures of the results are shown at the end of this report.

6 Conclusions

The objective of this study was to apply new, more detailed boundary conditions to the transient calculation of the load rejection test performed in 1998 on Olkiluoto 1 nuclear reactor. The exact boundary conditions for the valve variable were determined from the new measurement data and they were used in the calculation. In addition, the sensitivity of the calculation in respect of the valve boundary condition was evaluated by trying to adjust calculated pressure to measured pressure by varying the boundary condition freely. At first, only the first second of the calculation was considered and second, first three seconds were taken into account. The results are compared to the measured data as well as the original calculation.

The results are shown in the figures at the end of this report. Using the exact boundary conditions the mismatch appears to be too rapid in order to create the rise in pressure observed in the beginning of the test. Therefore, the new, exact boundary conditions do not enhance accuracy but rather impair it at the beginning of the calculation. Overall, this has little impact on the calculation as a whole. The calculations in which pressure was optimized for one and three seconds, are somewhat more accurate than the original calculation at the beginning, but their influence lasts no more than a couple of second. The impact of the new boundary conditions is very little on other variables apart from pressure and steam flow out of steam line.

As a conclusion, it can be summarized that the boundary conditions on the valve variable are not critical concerning the load rejection test calculation. Some enhancements can be attained during the first few seconds of the calculation, but in order to do so the values of the valve variable must be changed substantially from the one gained from measured data. Even when doing so, no effect is seen after four seconds.

In the future, improvements on the calculation of the load rejection test could be attained for example by adding a feed water controller model to the input. In the

present calculations height of water level is fairly accurate until 100 seconds. After that, however, the absence of the controller model shows evidently.

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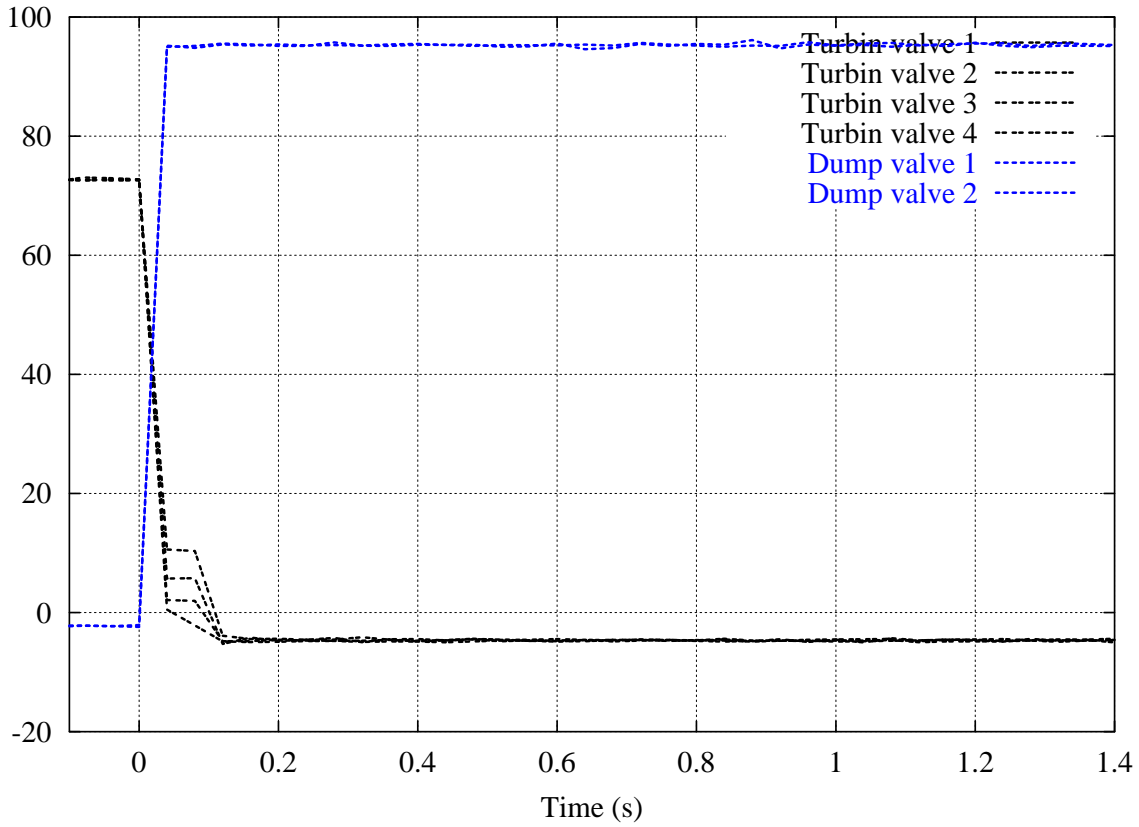


Figure 2. Measured data of turbine and dump valves.

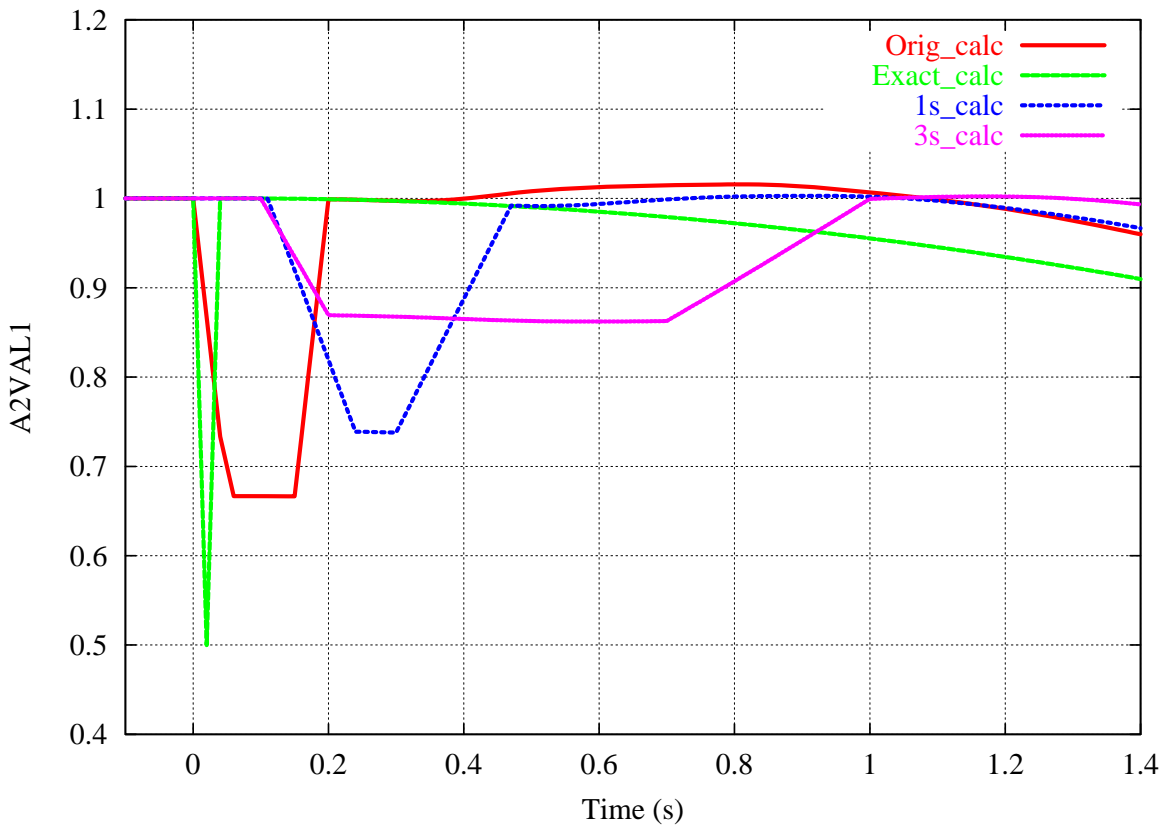


Figure 3. Valve boundary condition in the original and new calculations.

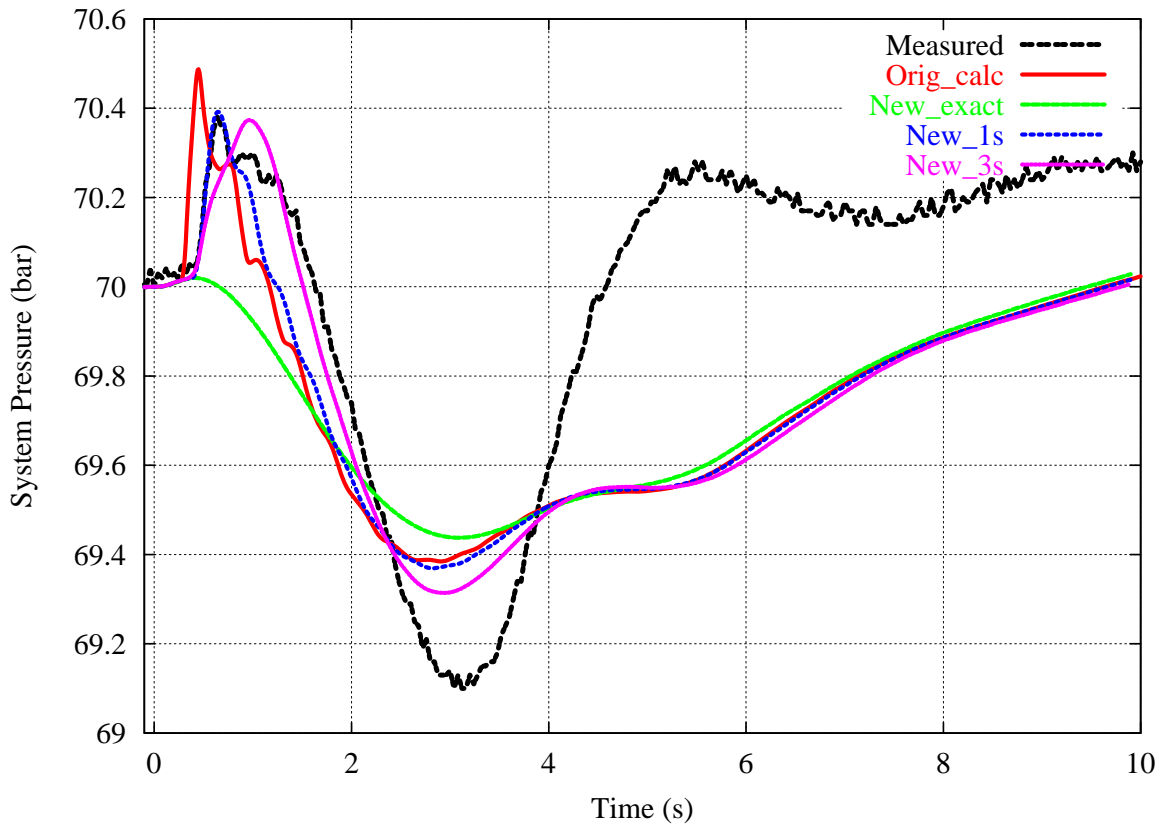


Figure 4. Measured and calculated system pressure.

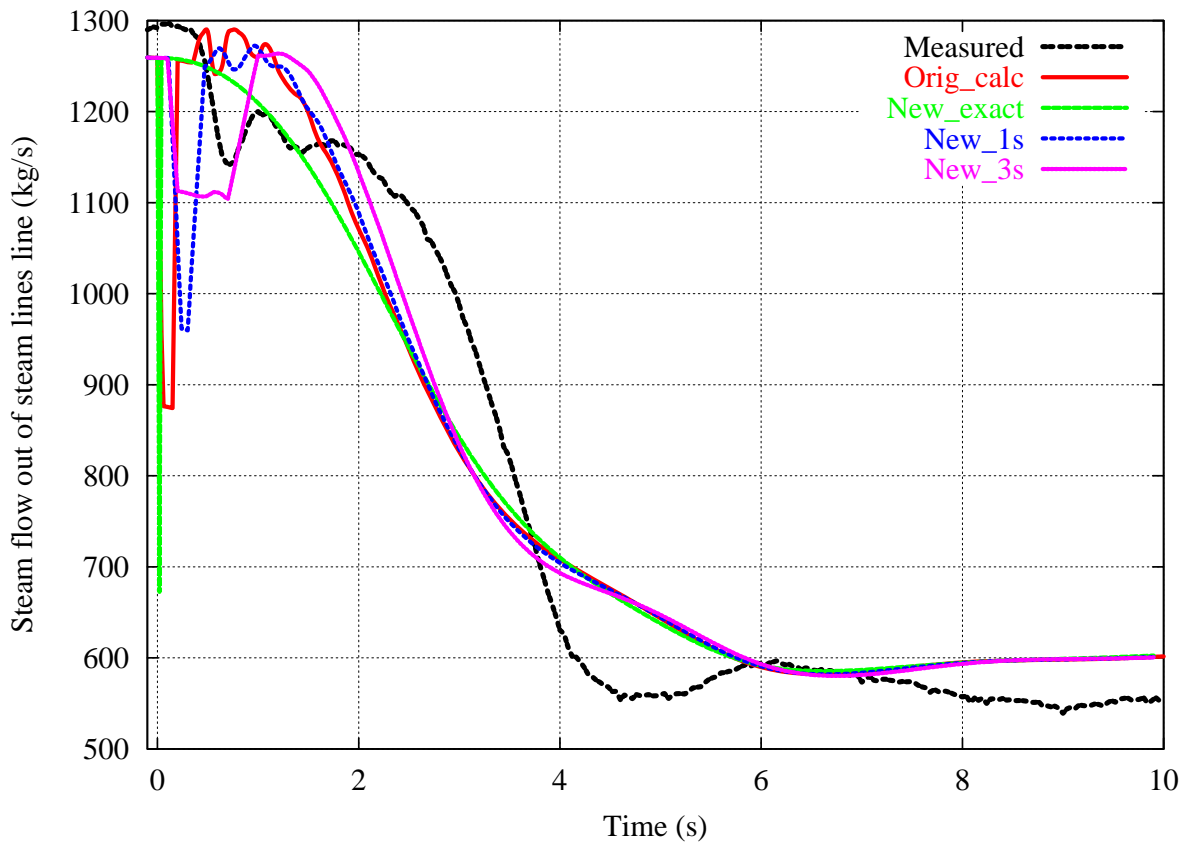


Figure 5. Measured and calculated steam flow out of steam line 1.

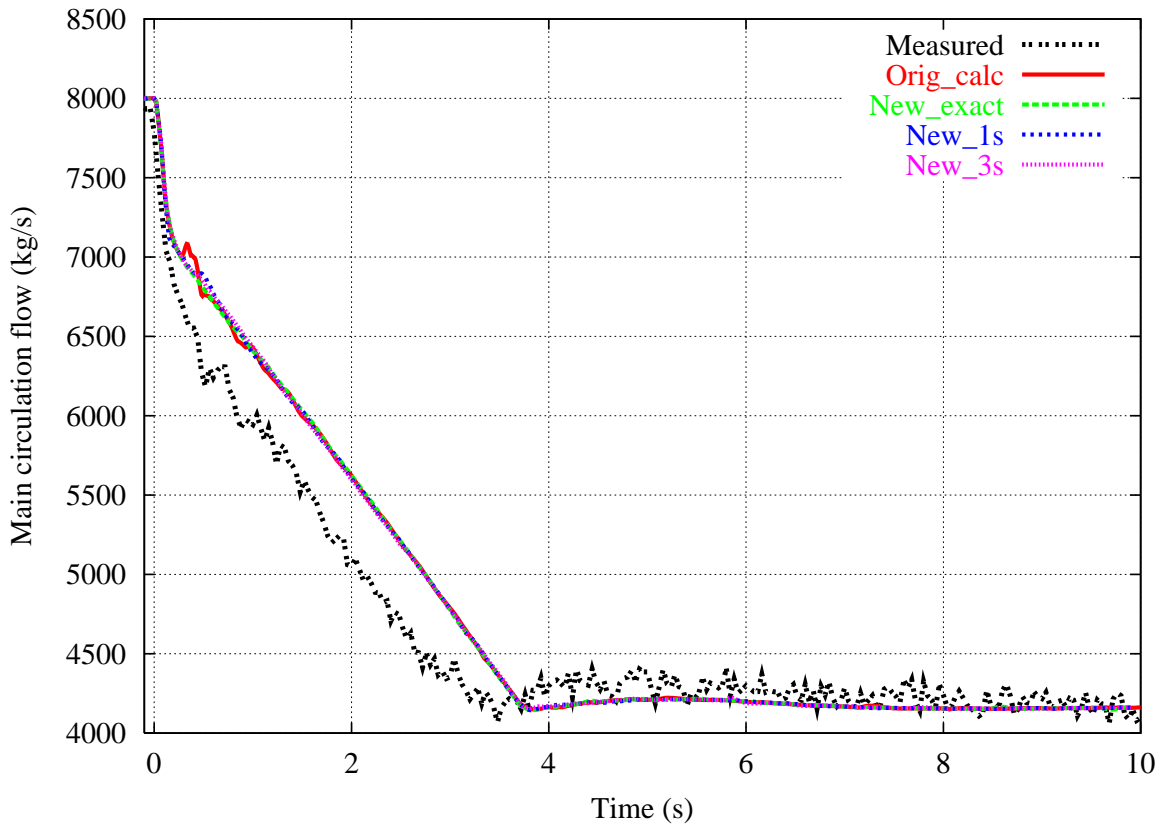


Figure 6. Measured and calculated main circulation flow.

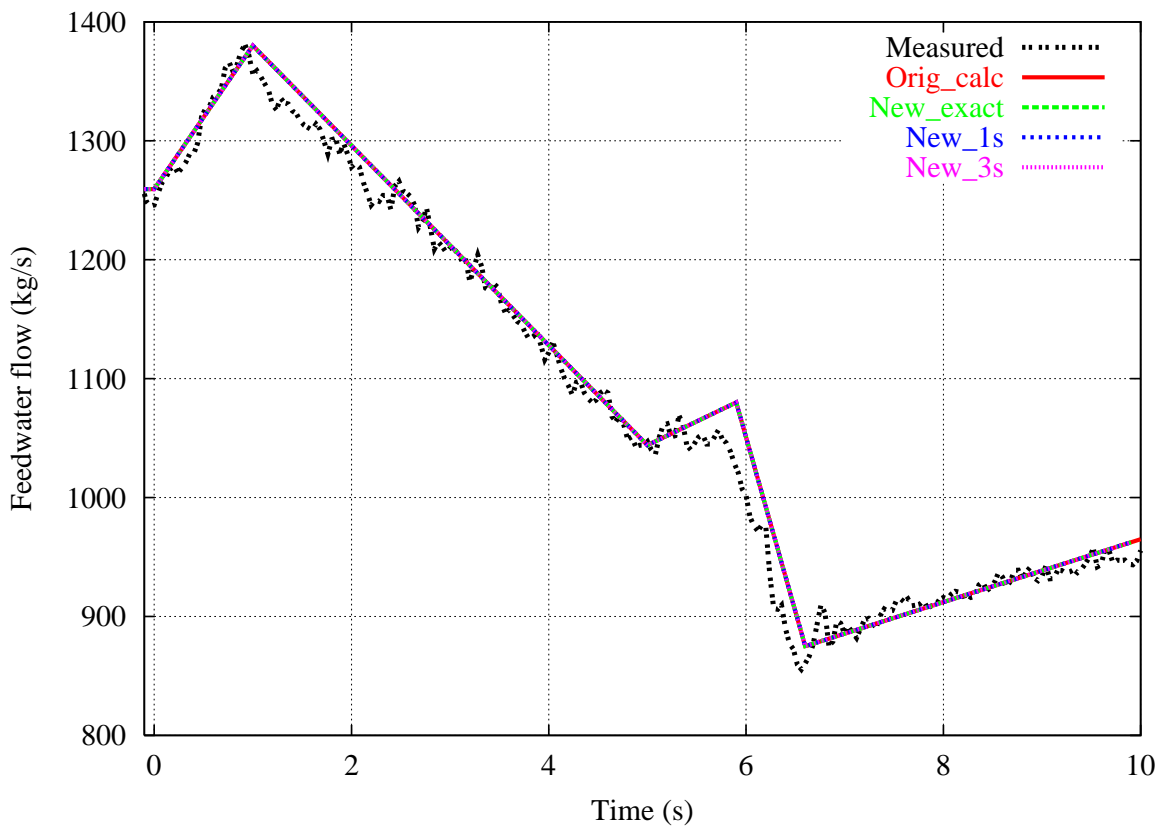


Figure 7. Measured and calculated feedwater flow. Boundary condition.

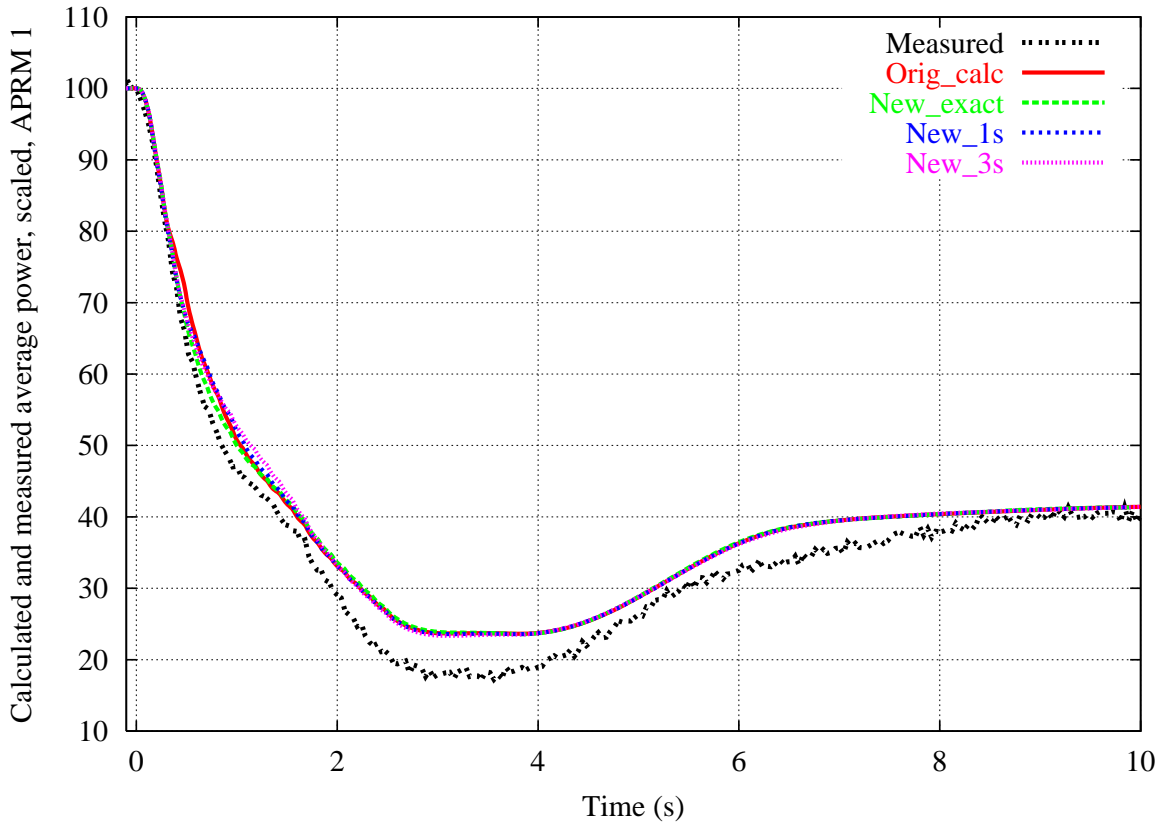


Figure 8. Calculated and measured average power, APRM 1.

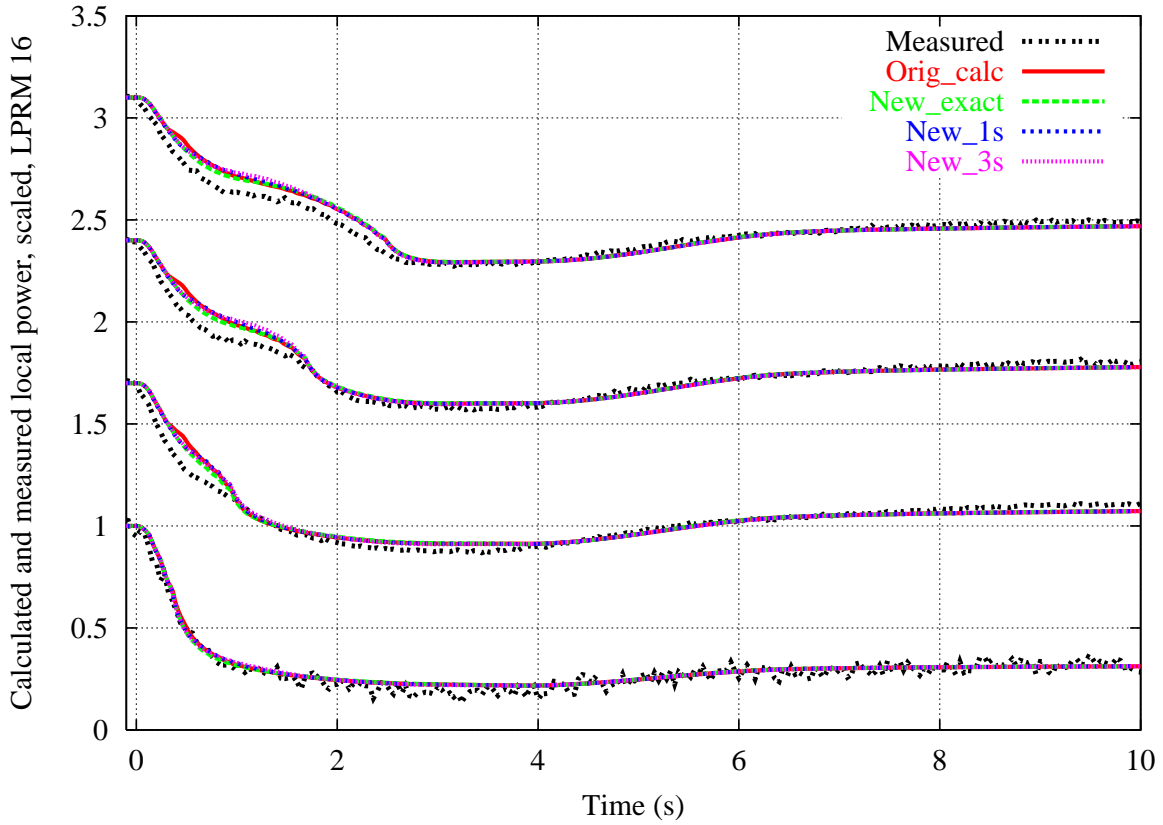
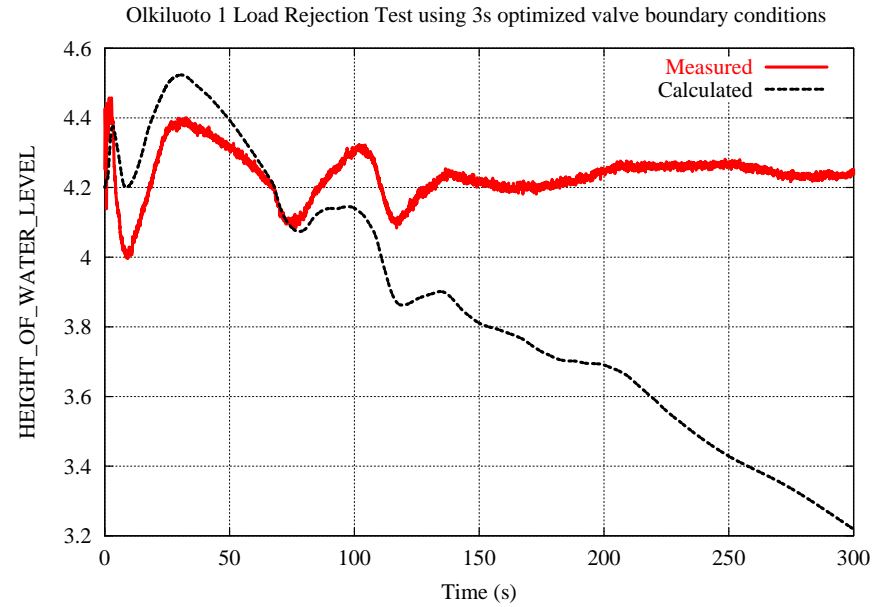
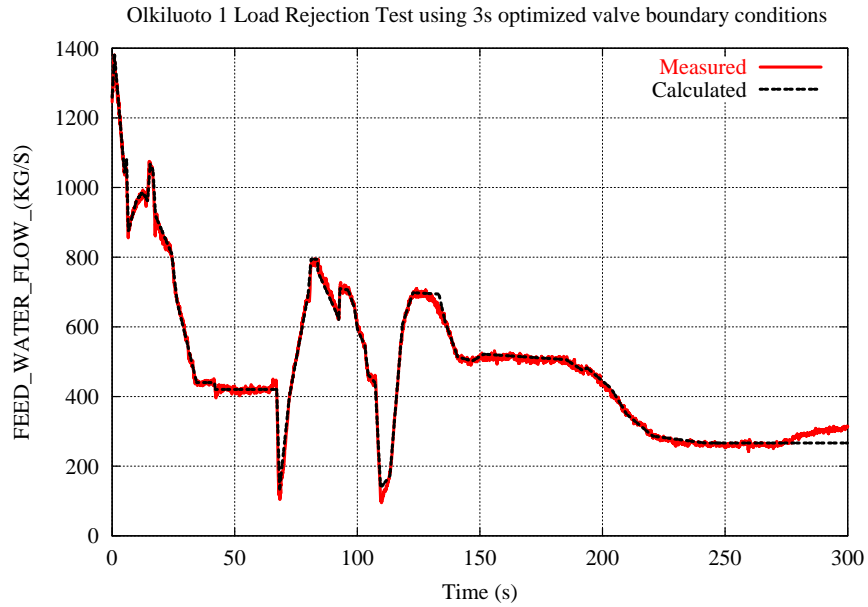
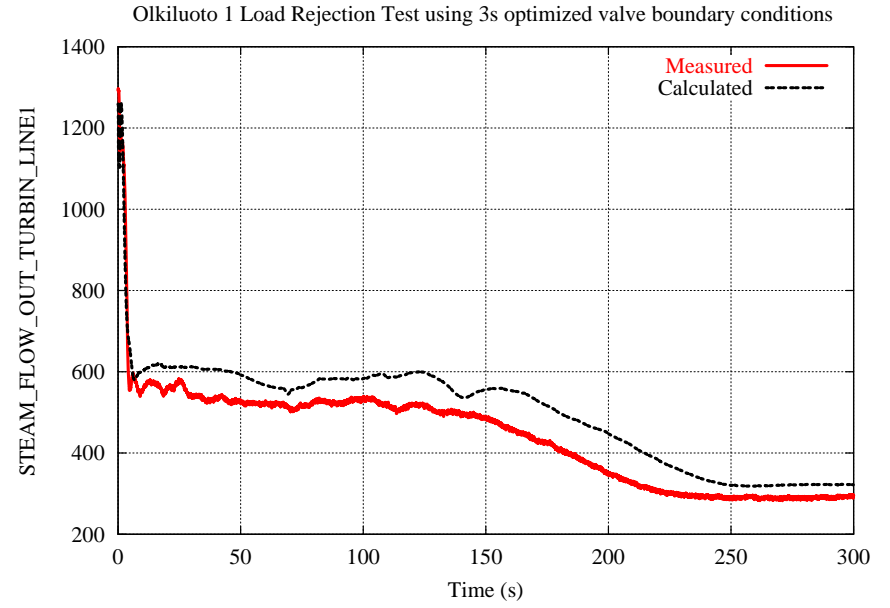
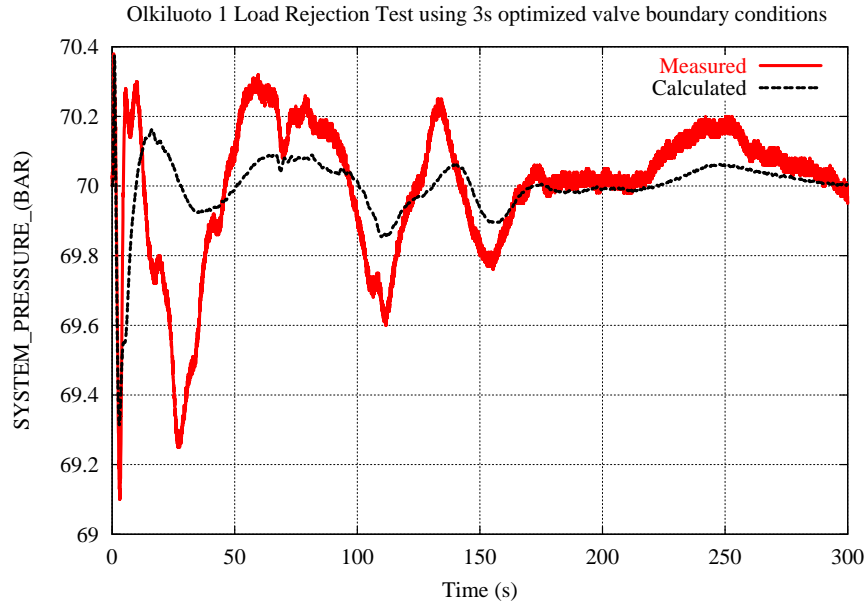
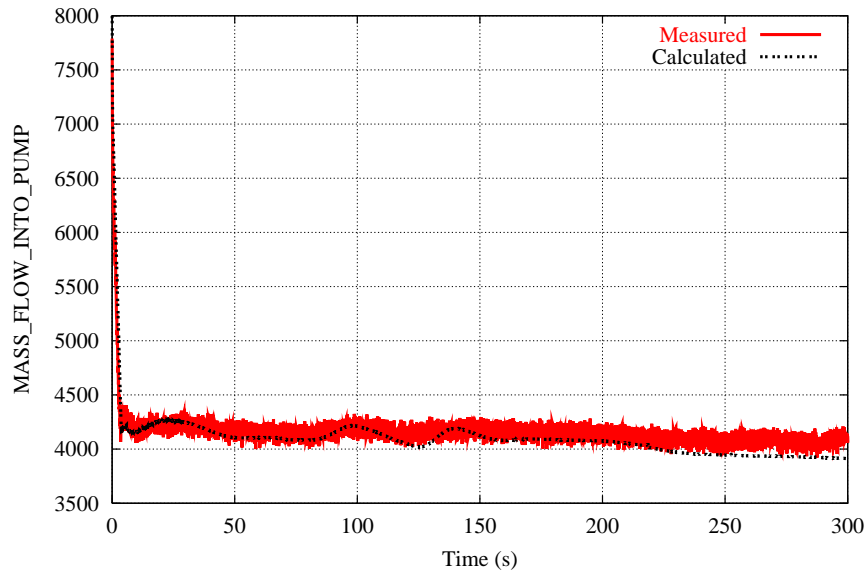


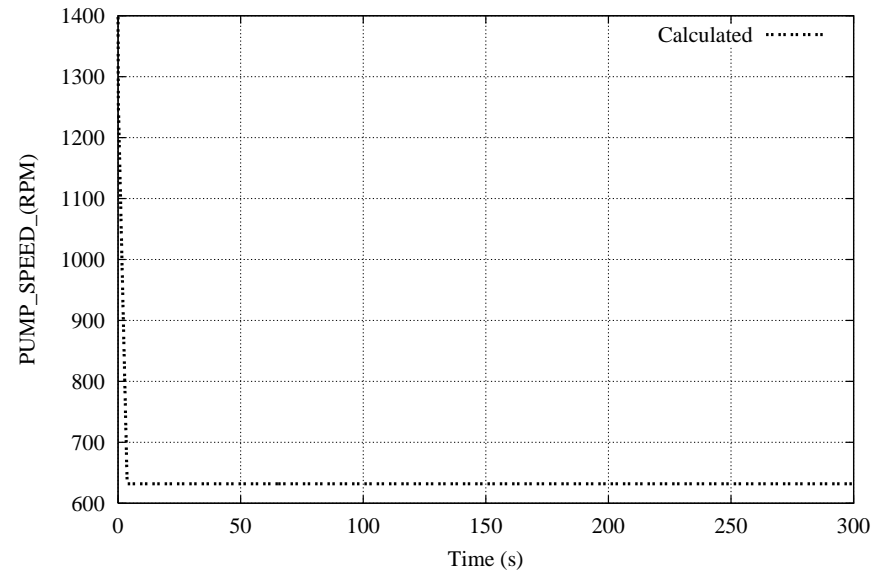
Figure 9. Calculated and measured local power, LPRM 16.



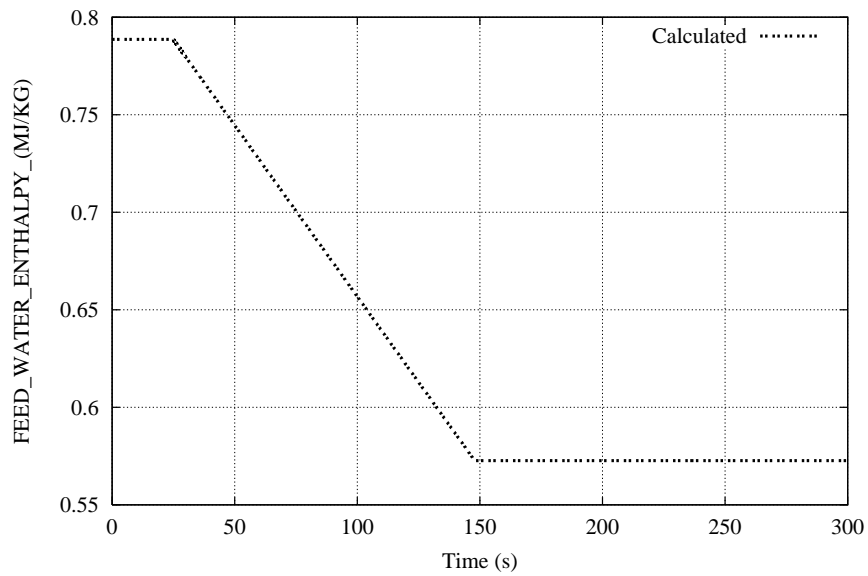
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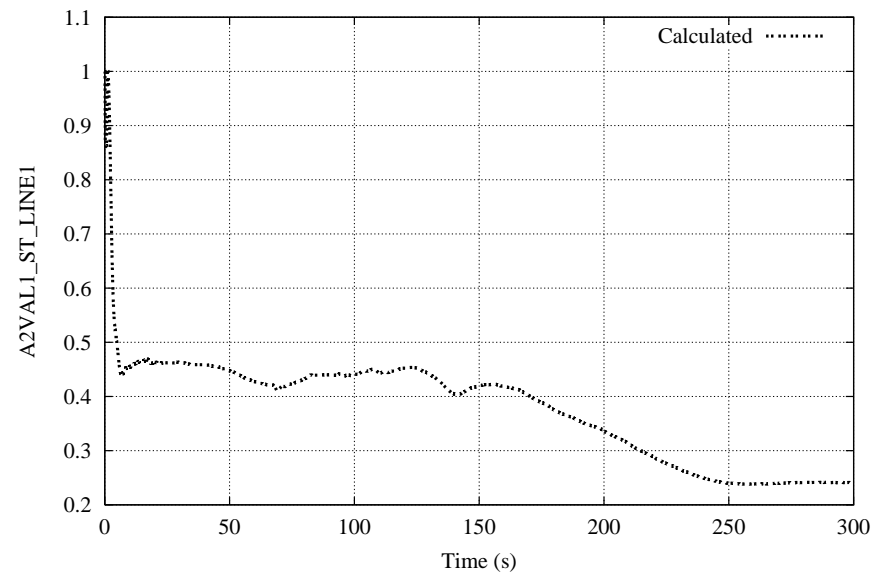
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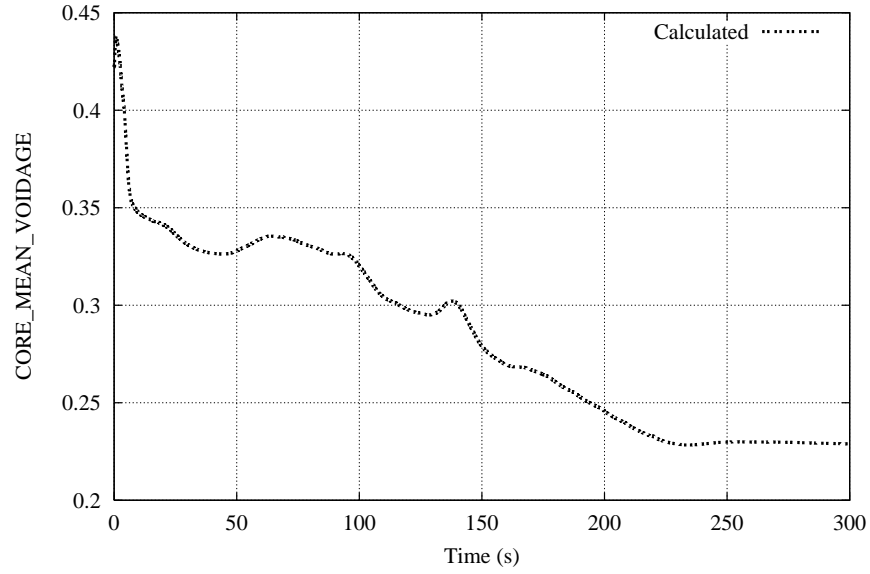
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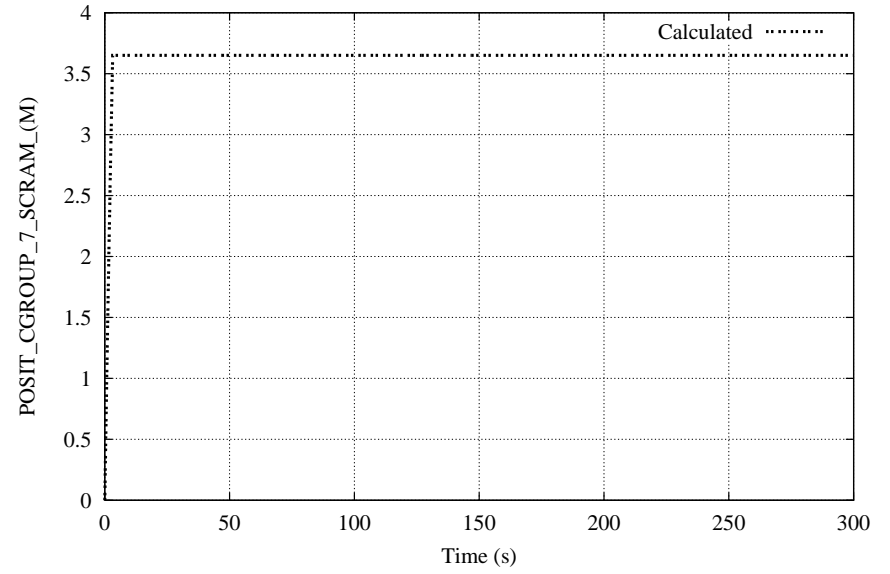
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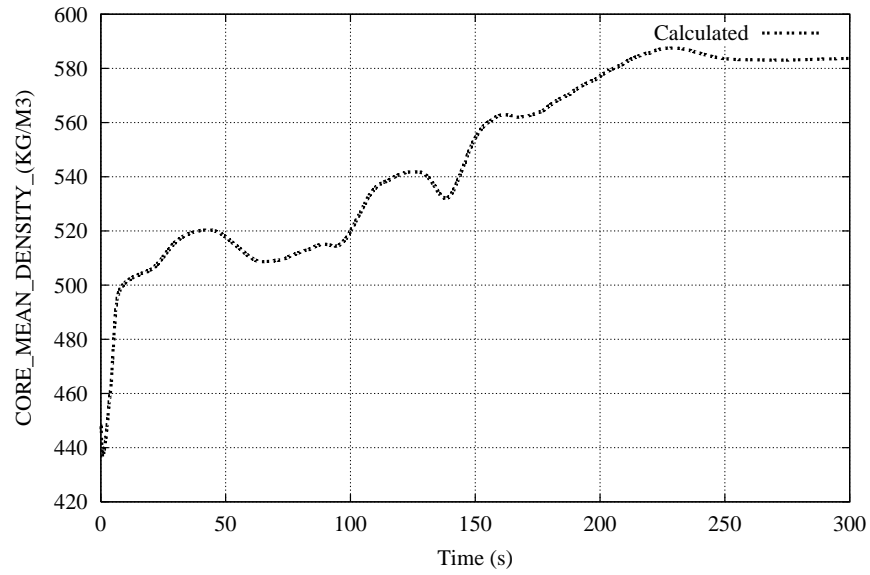
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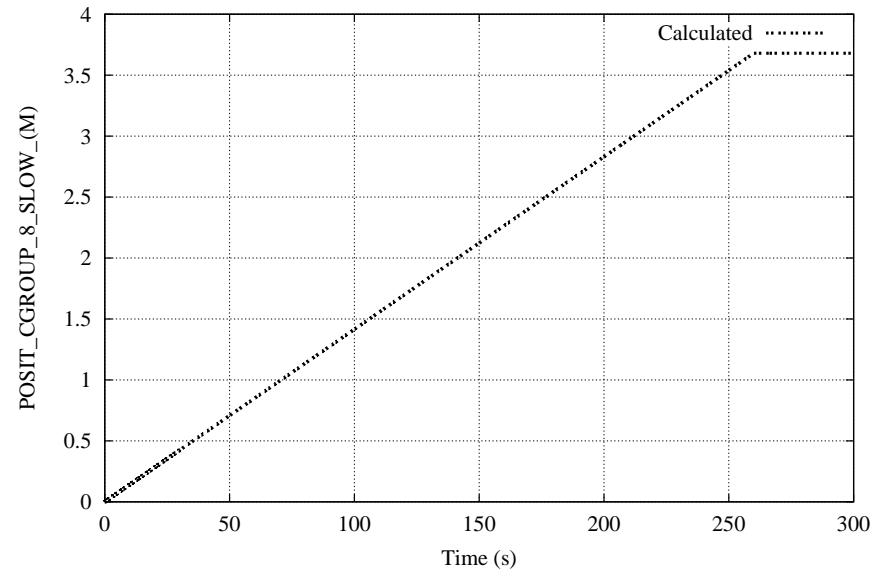
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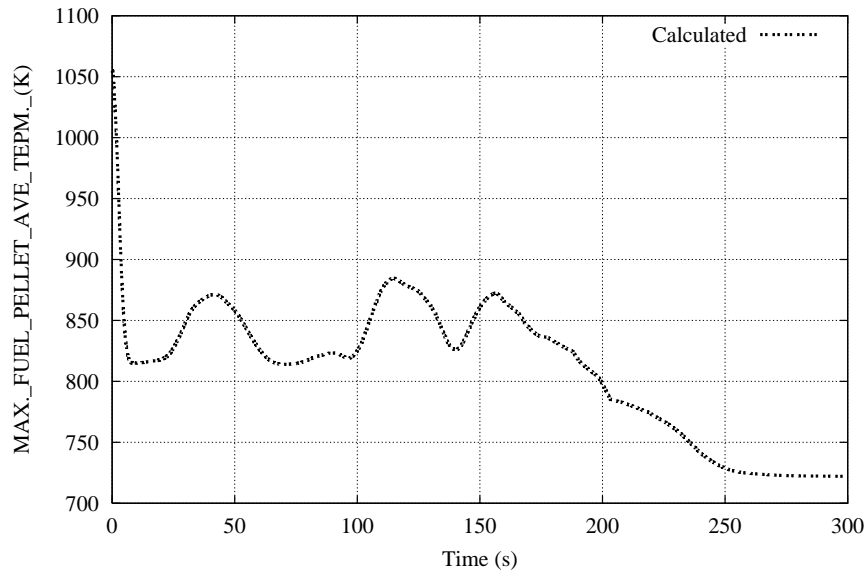
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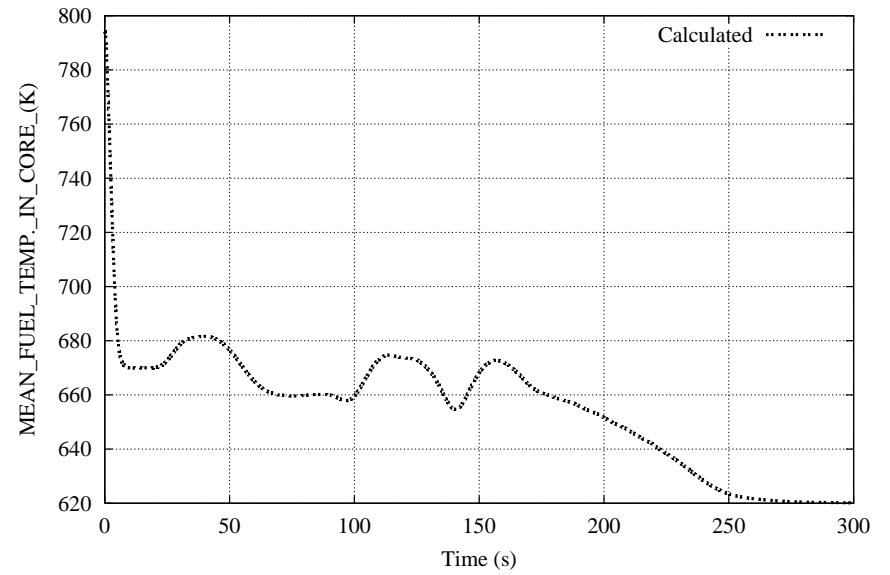
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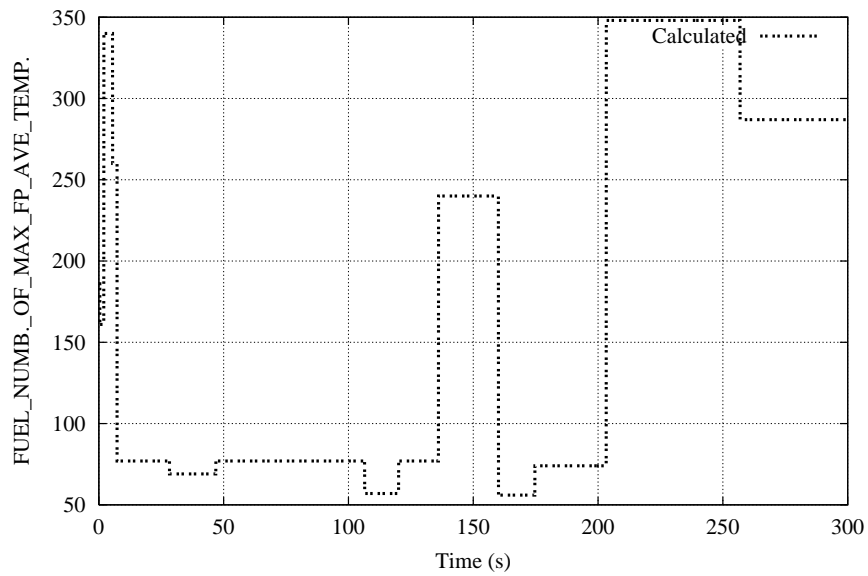
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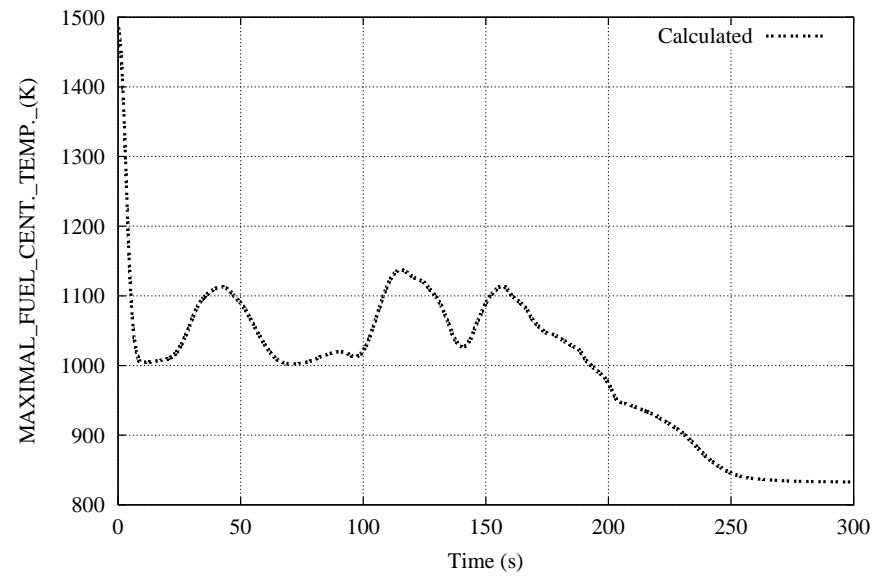
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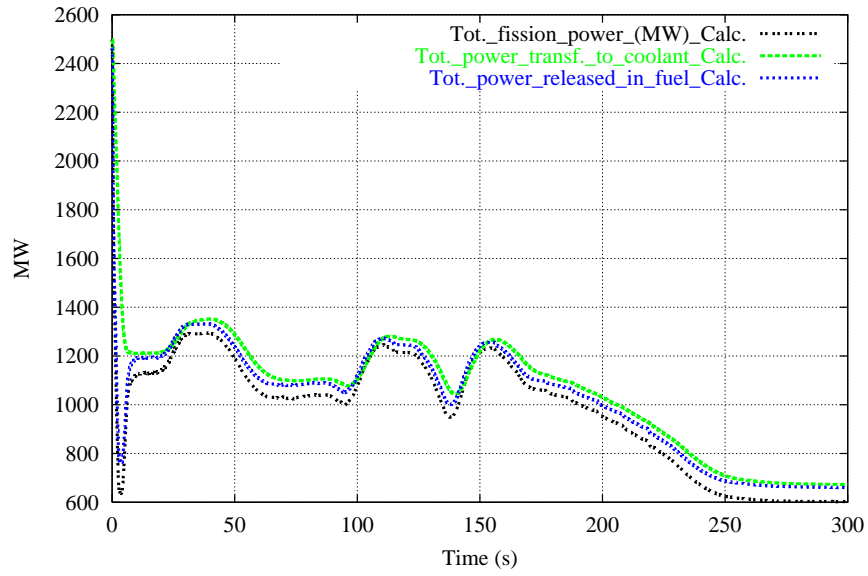
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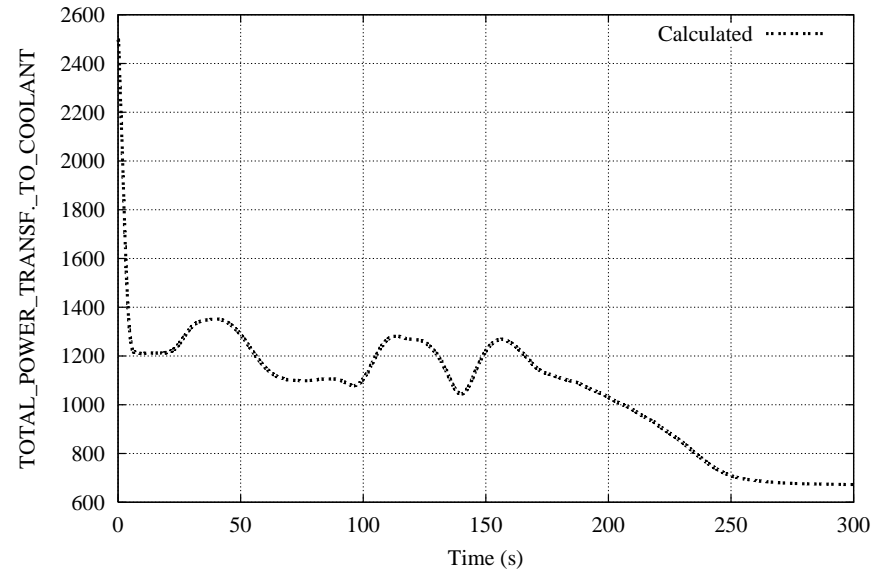
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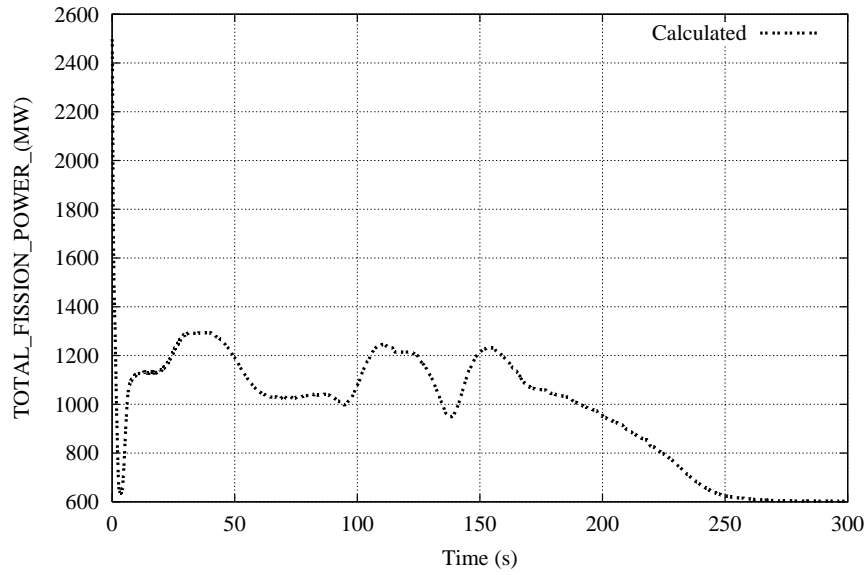
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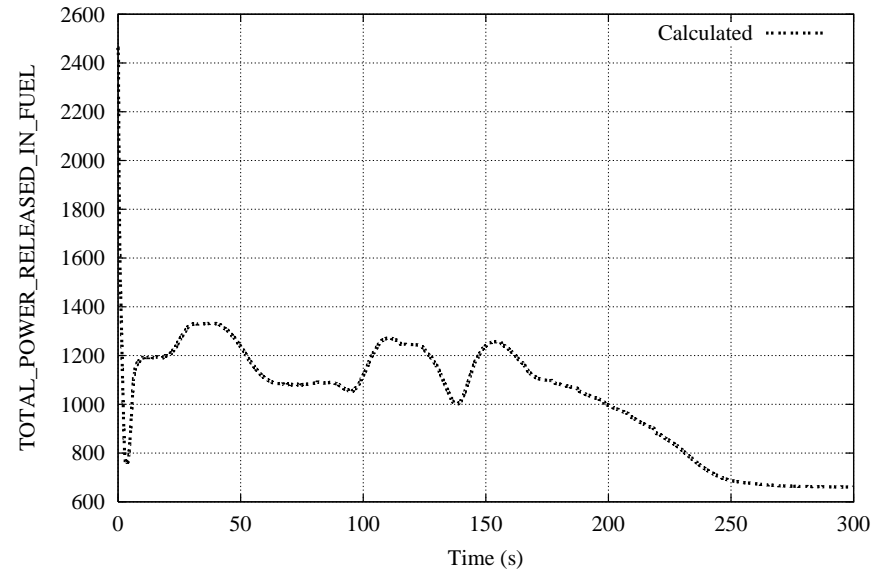
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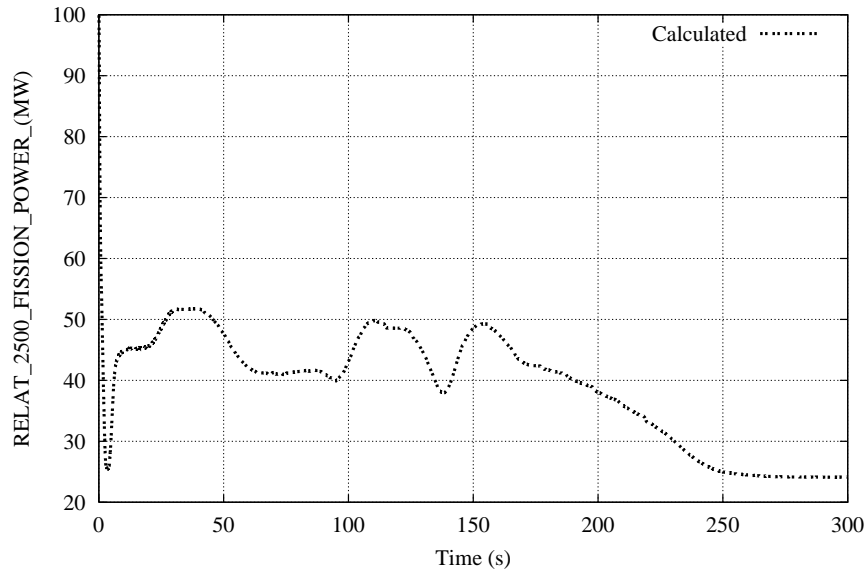
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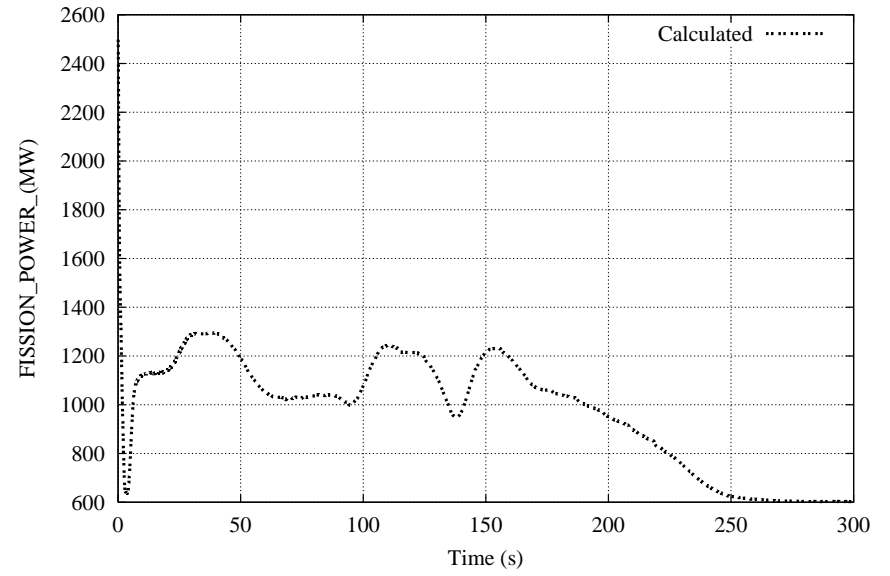
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



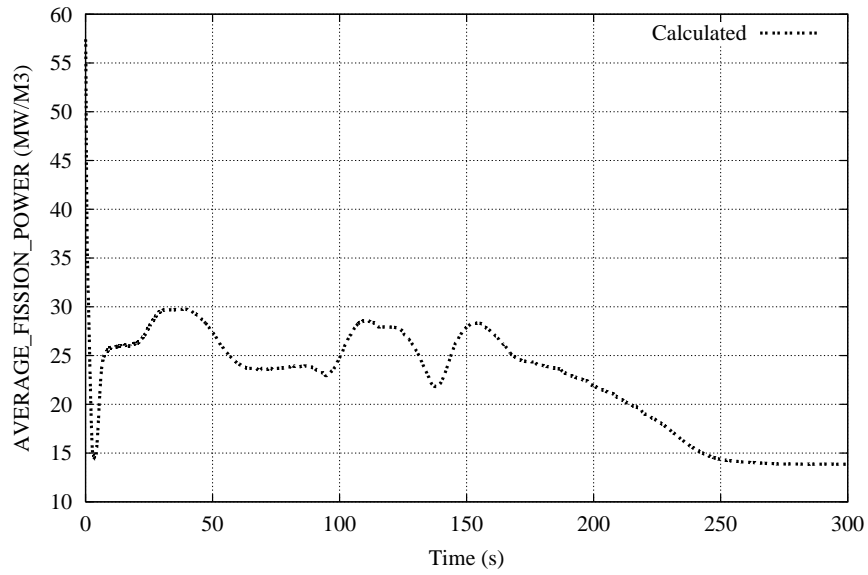
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



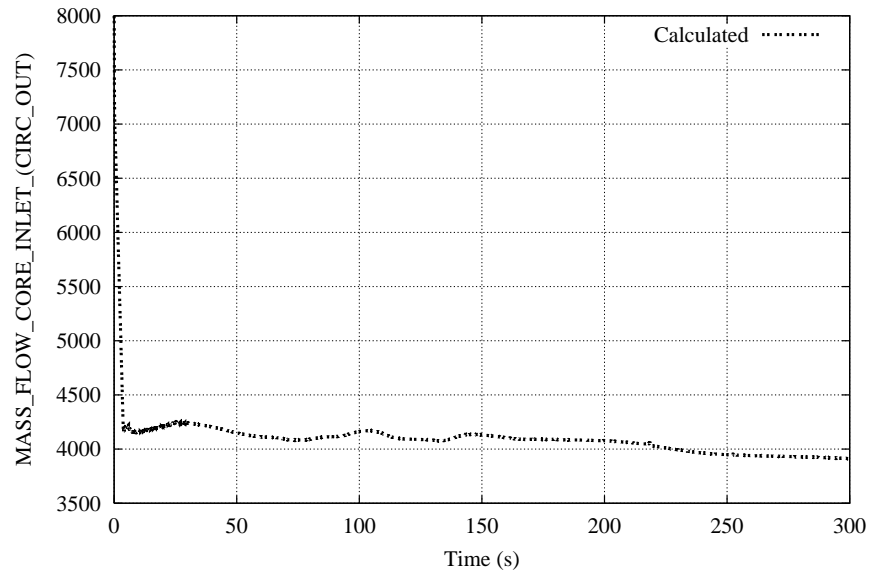
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



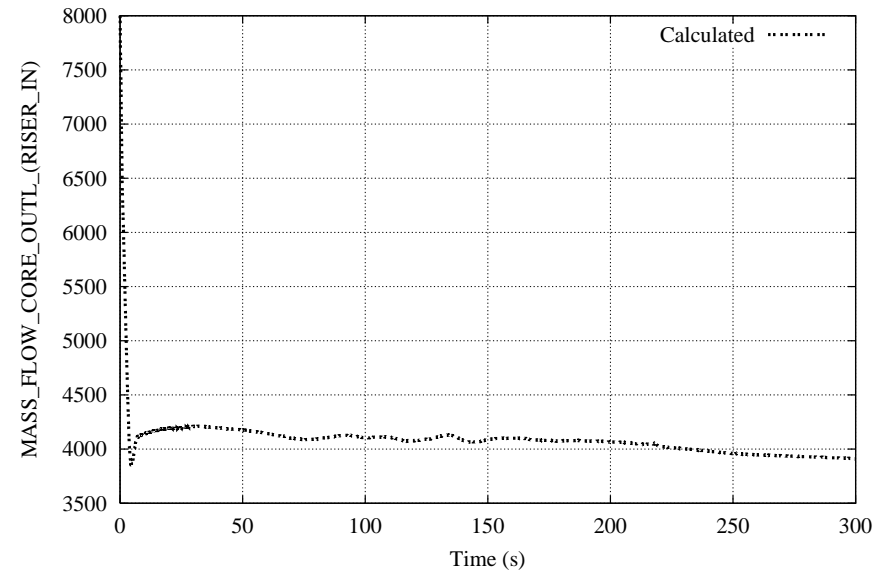
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



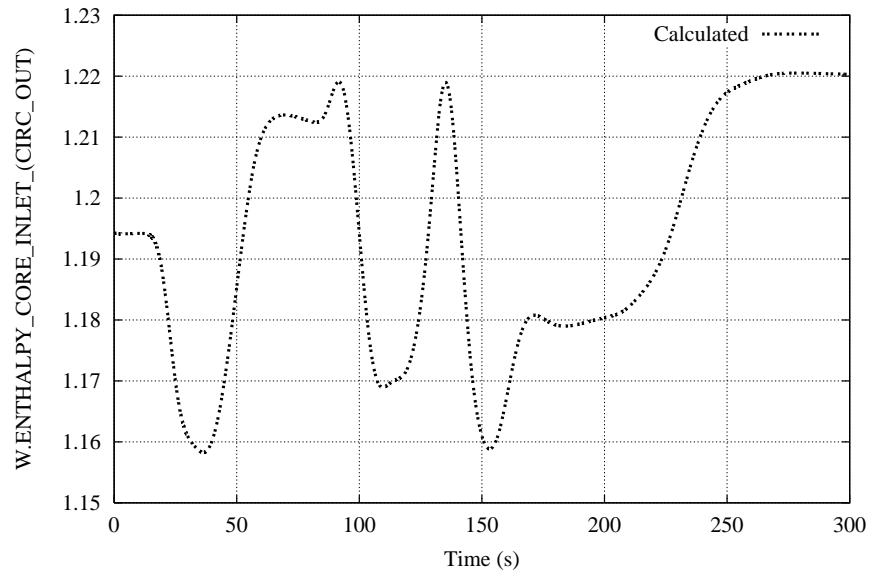
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



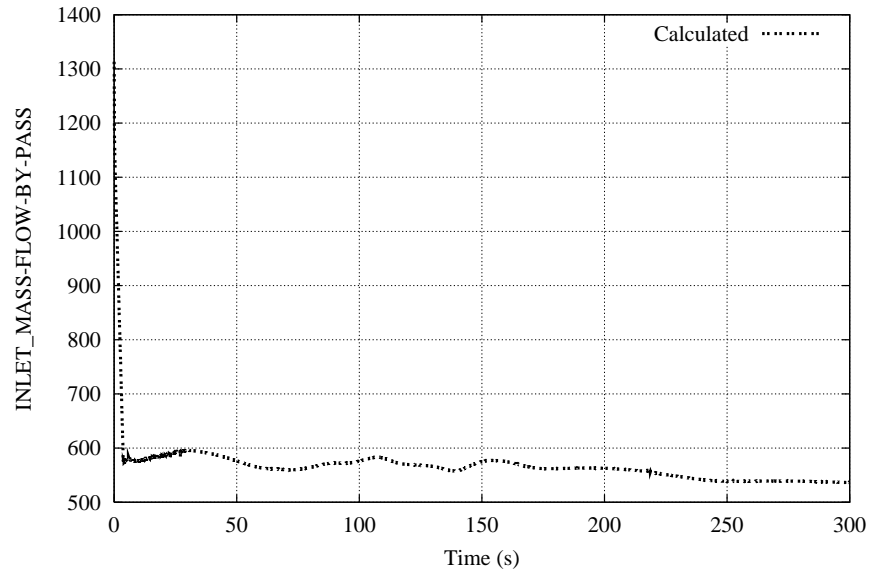
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



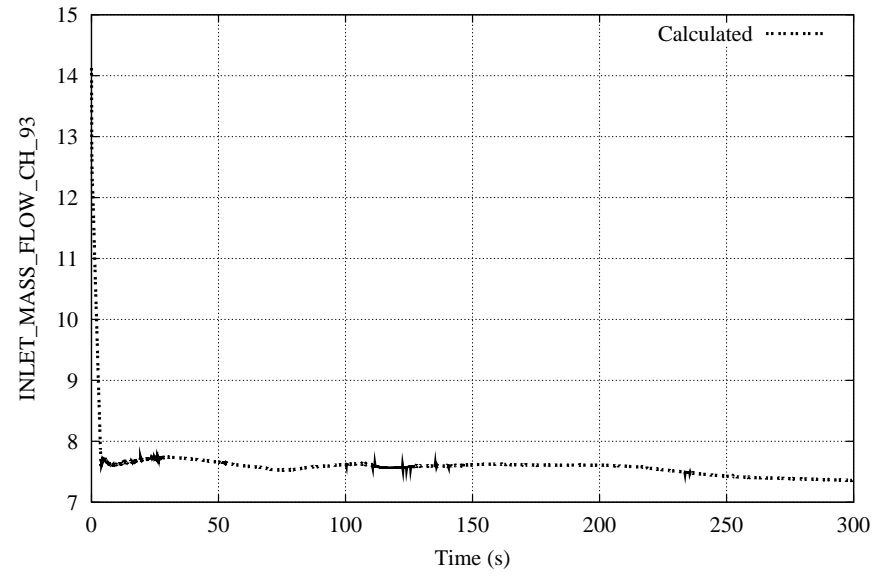
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



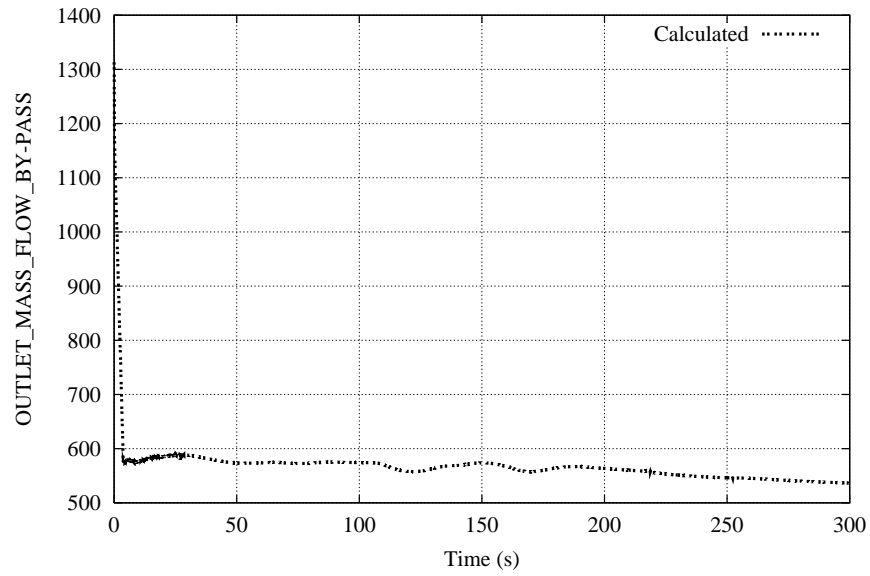
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



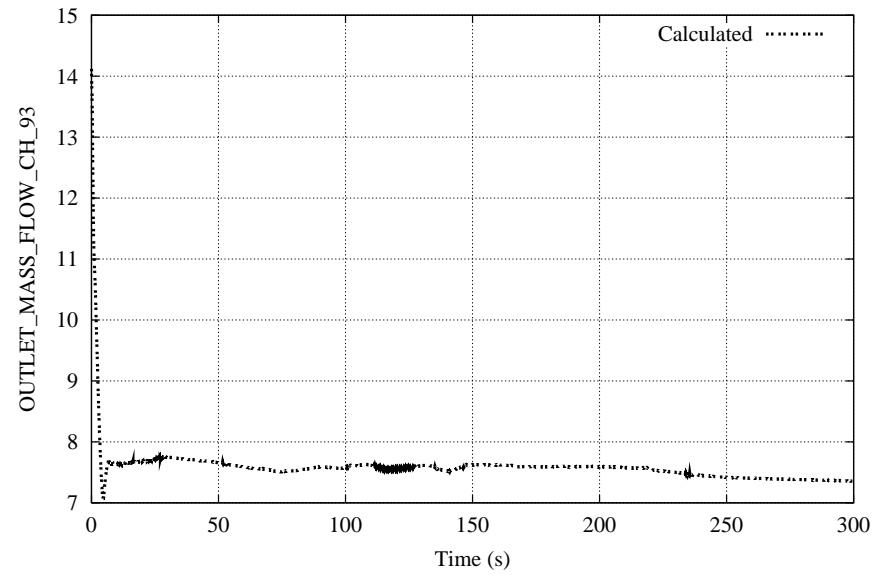
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



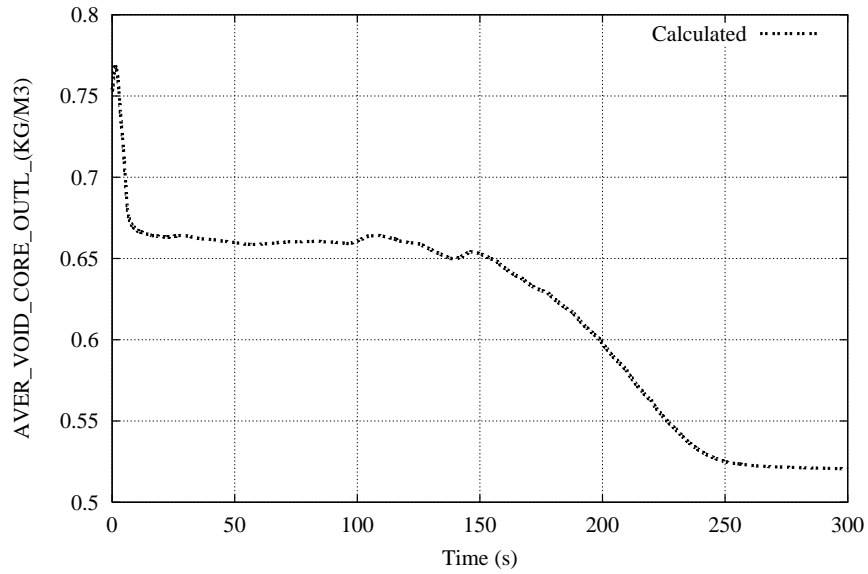
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



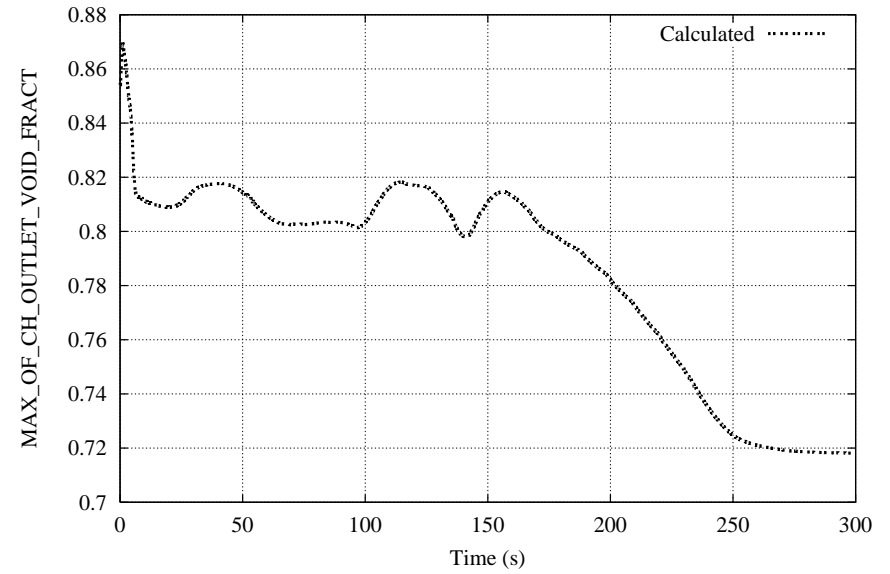
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



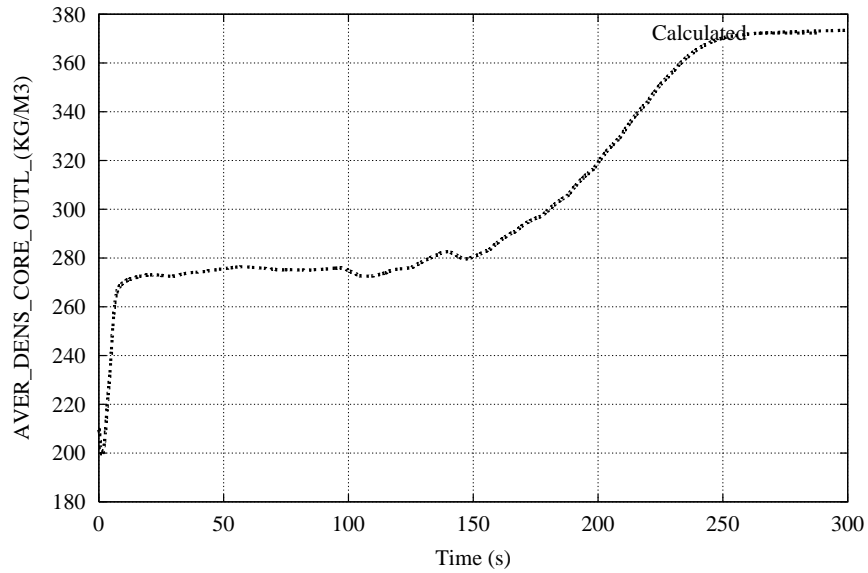
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



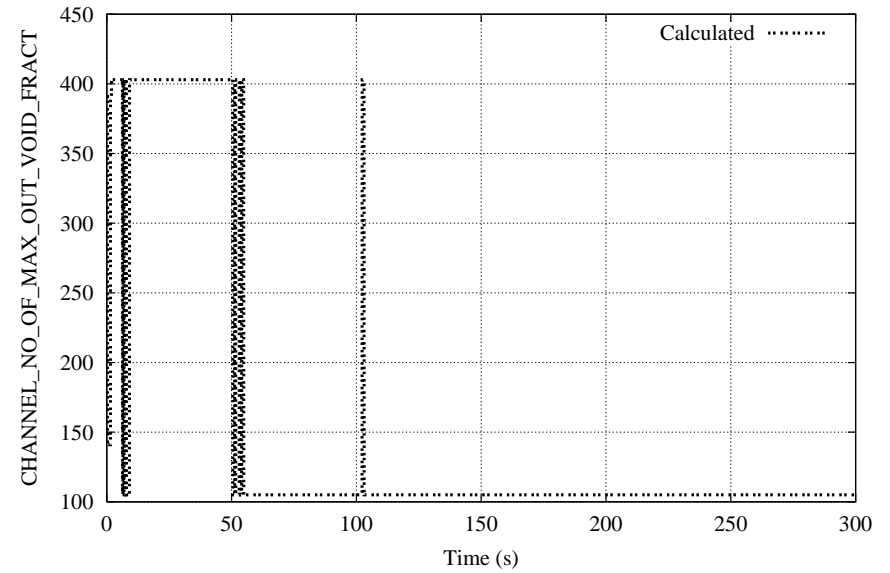
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



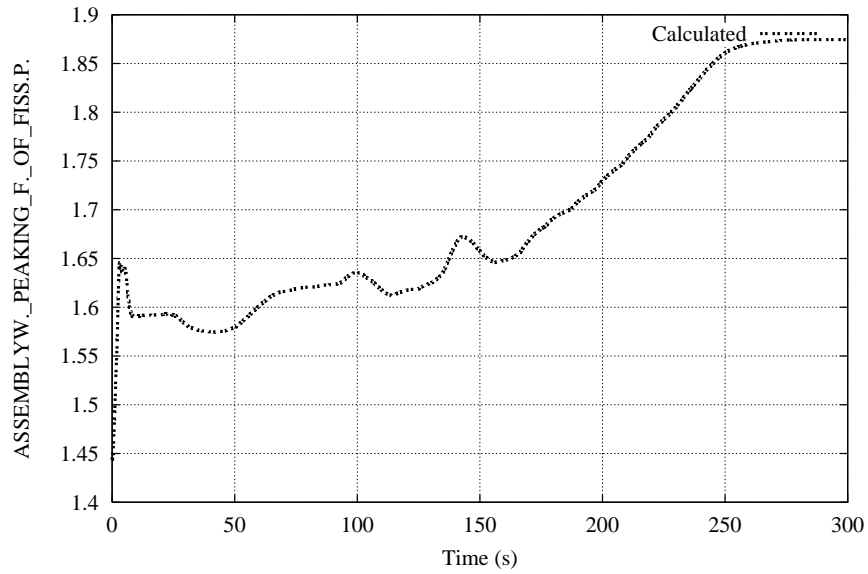
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



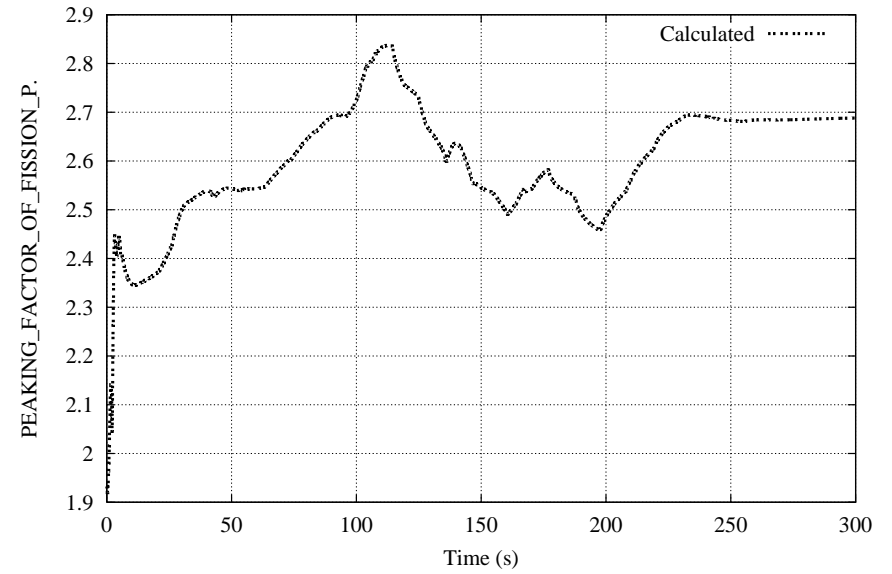
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



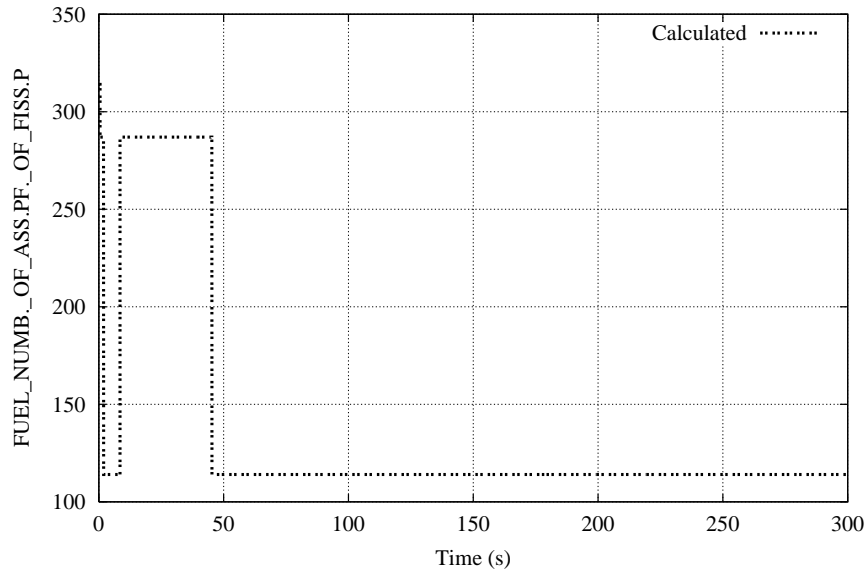
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



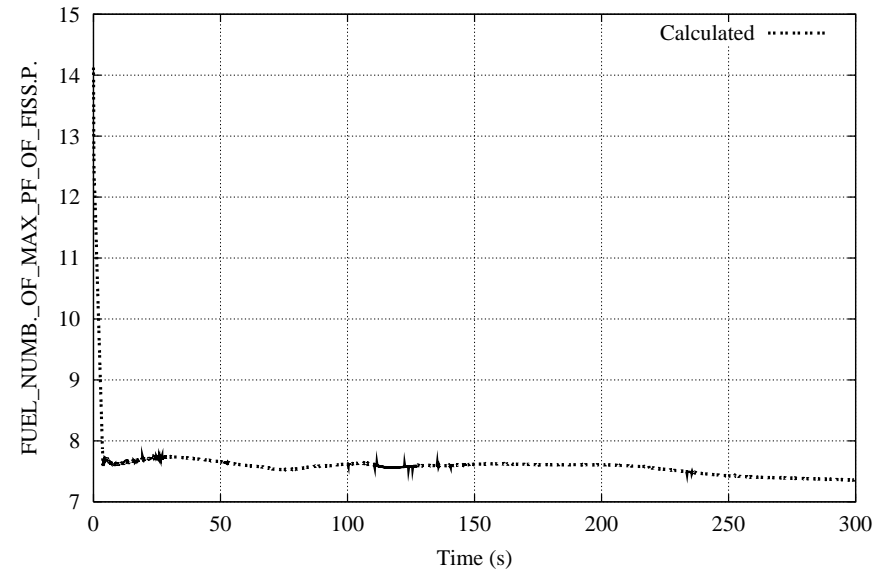
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



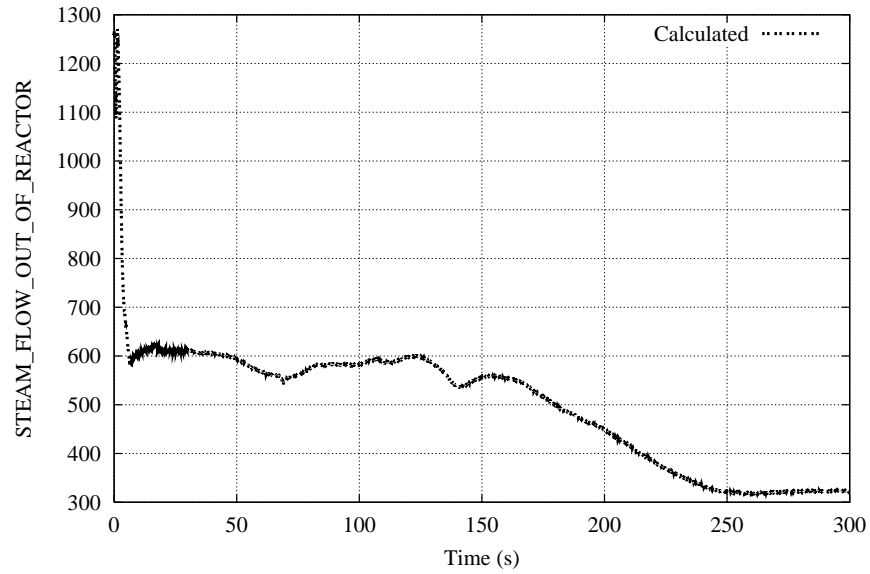
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



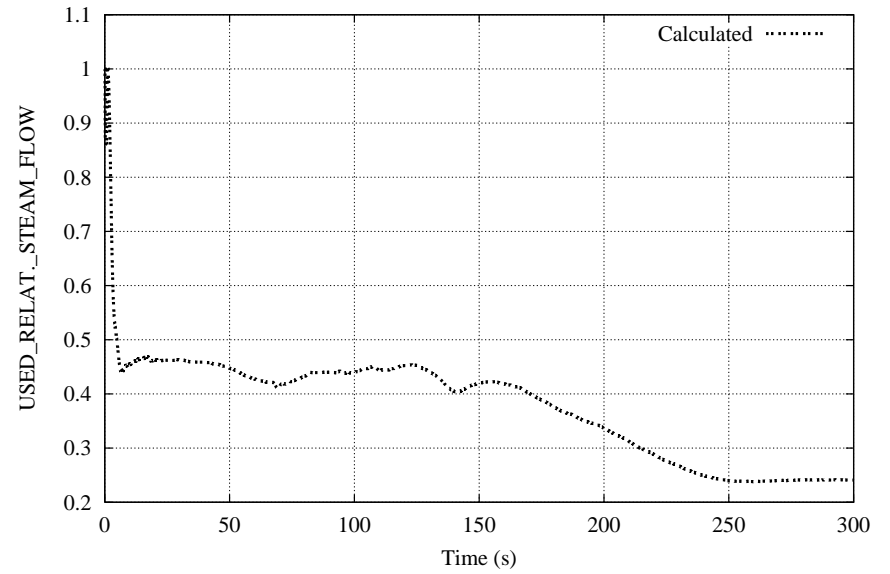
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



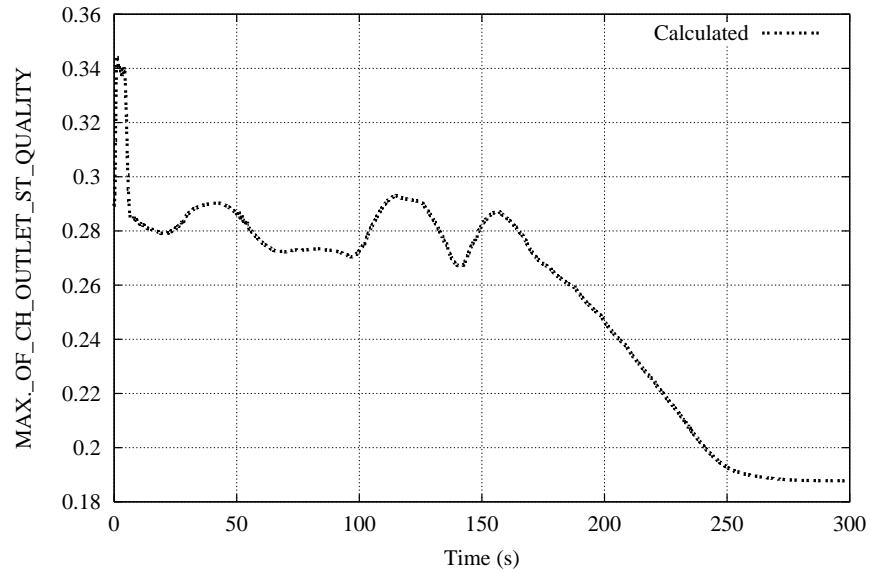
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



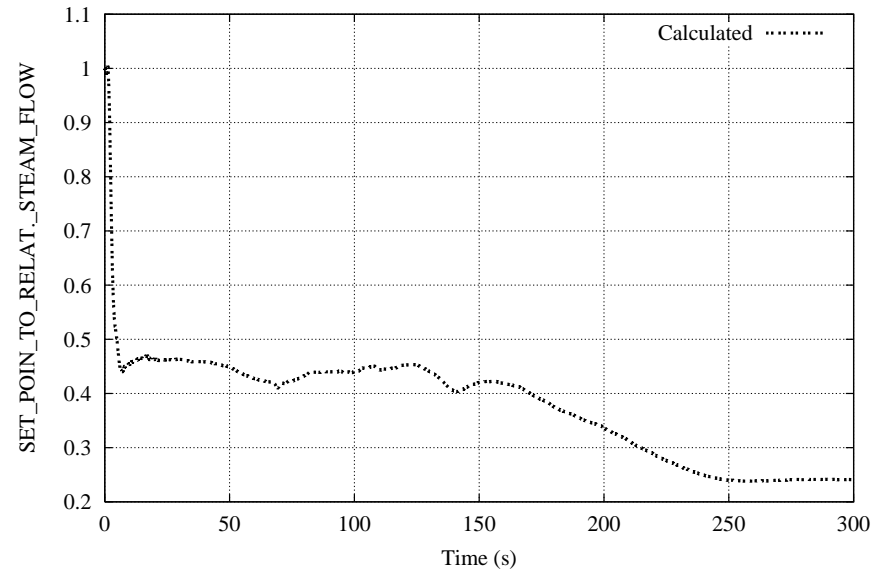
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



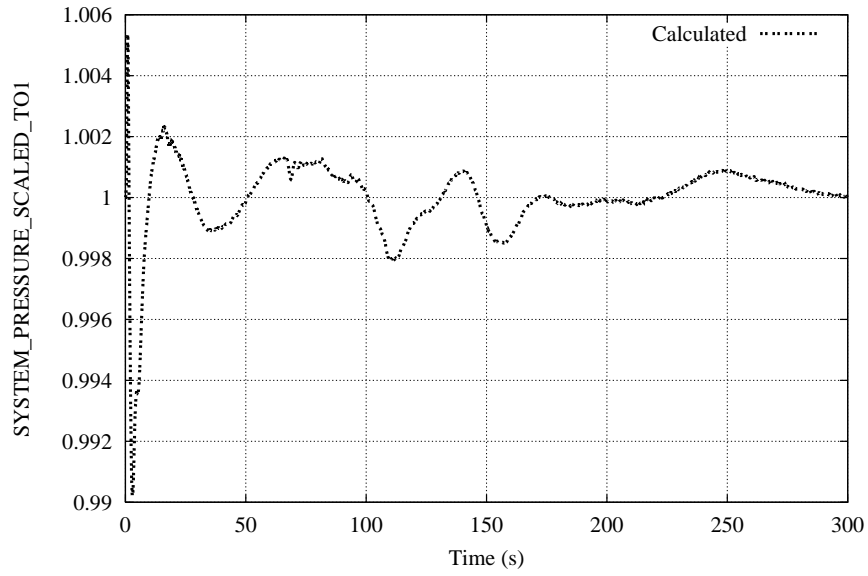
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



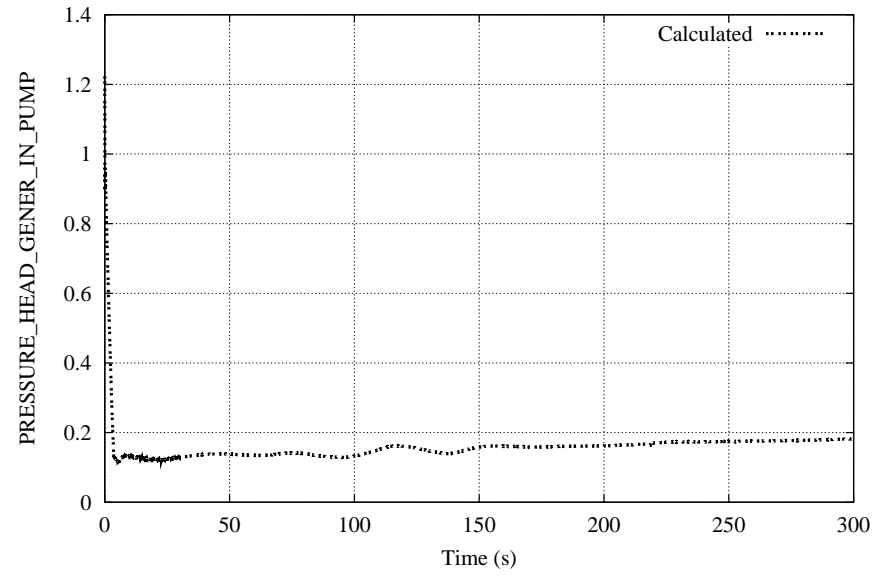
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



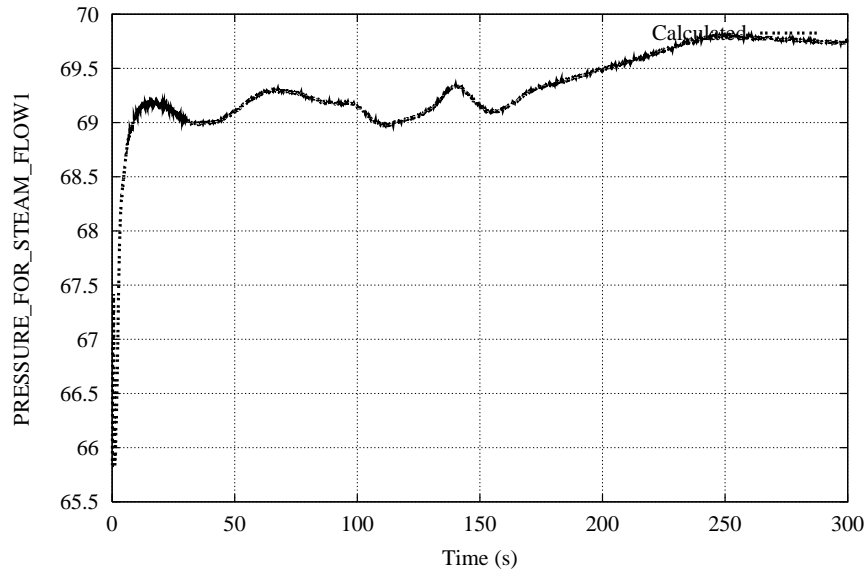
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



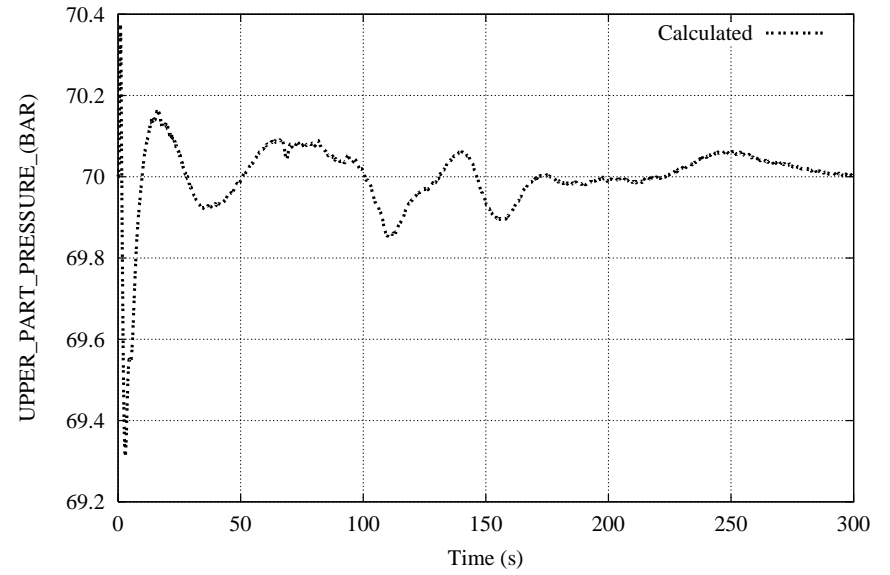
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



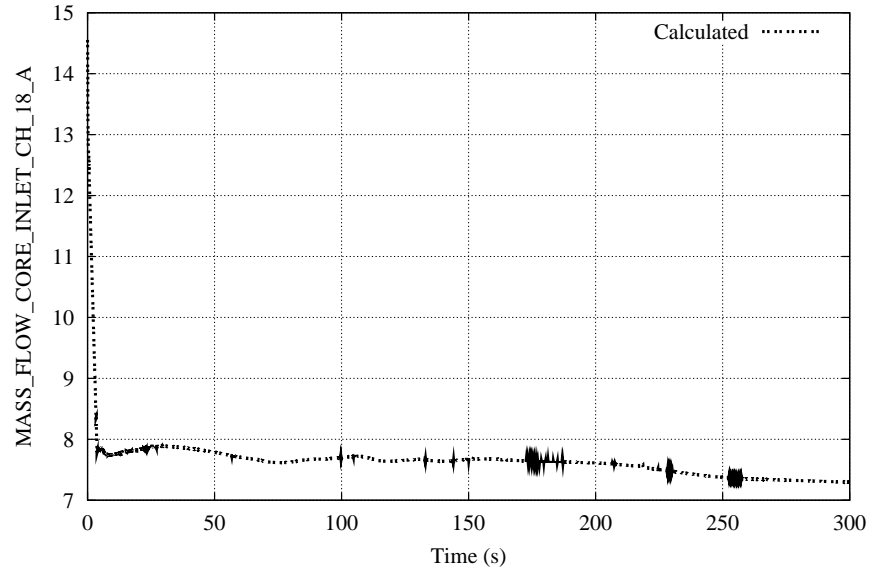
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



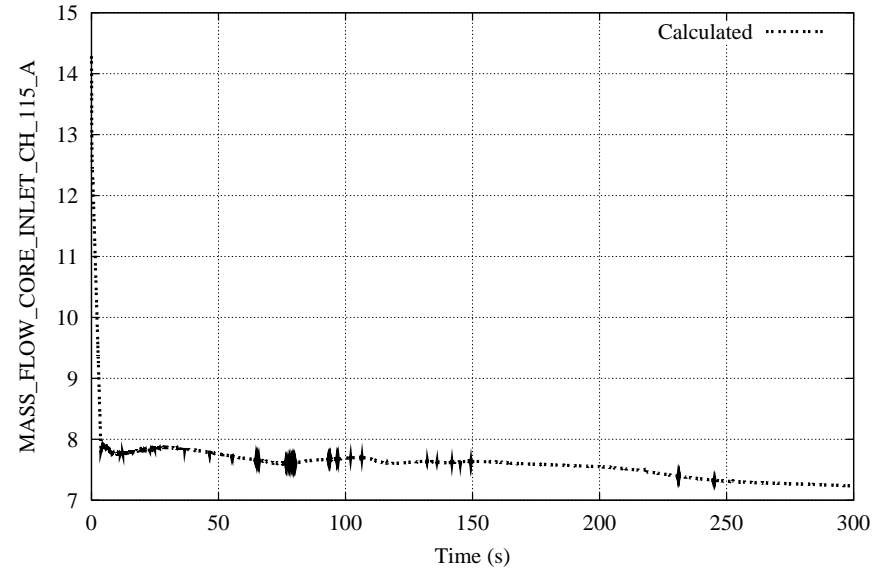
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



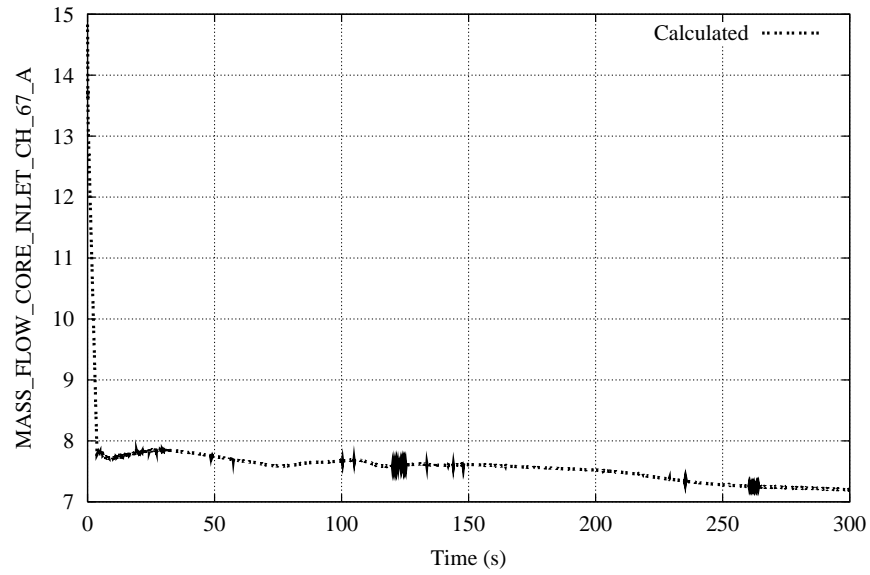
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



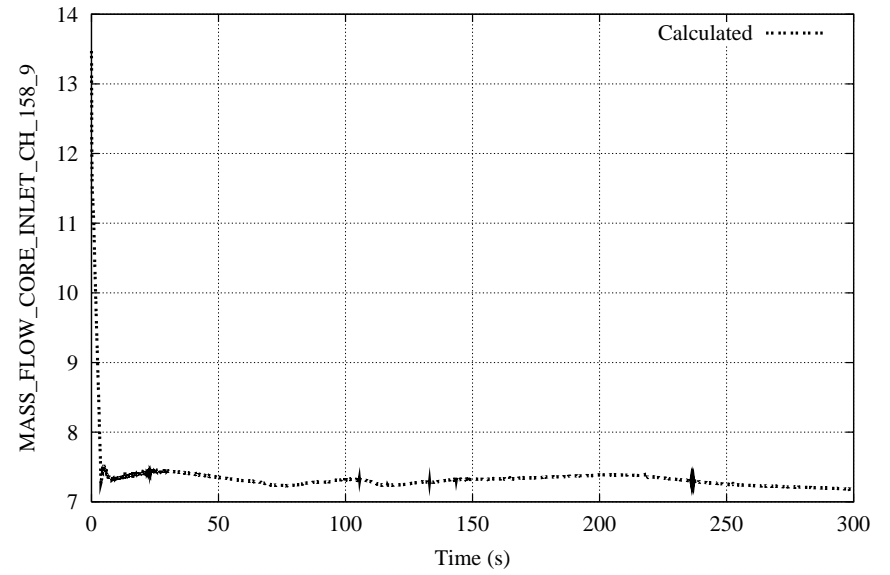
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



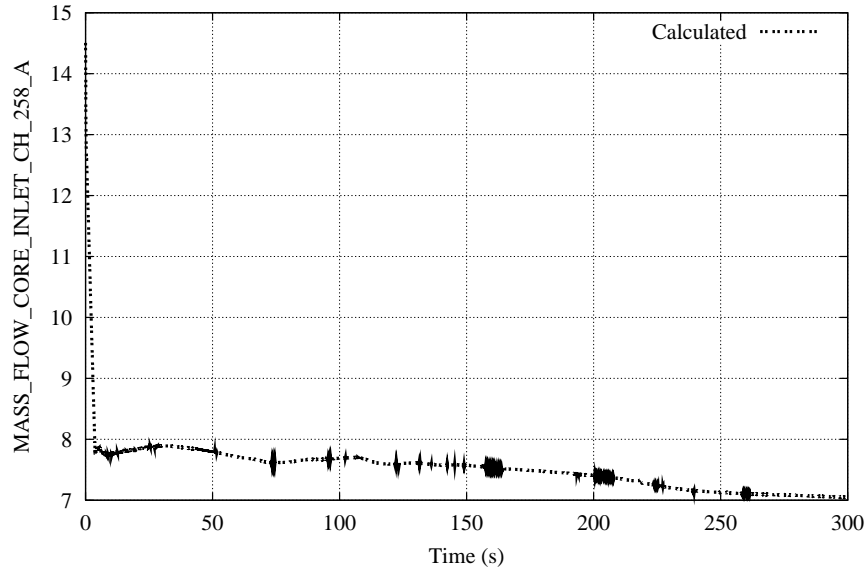
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



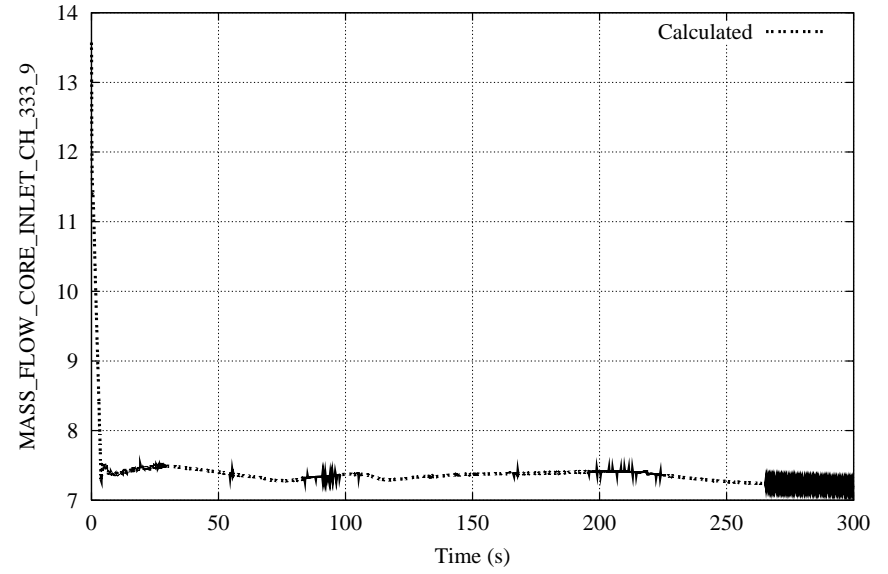
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



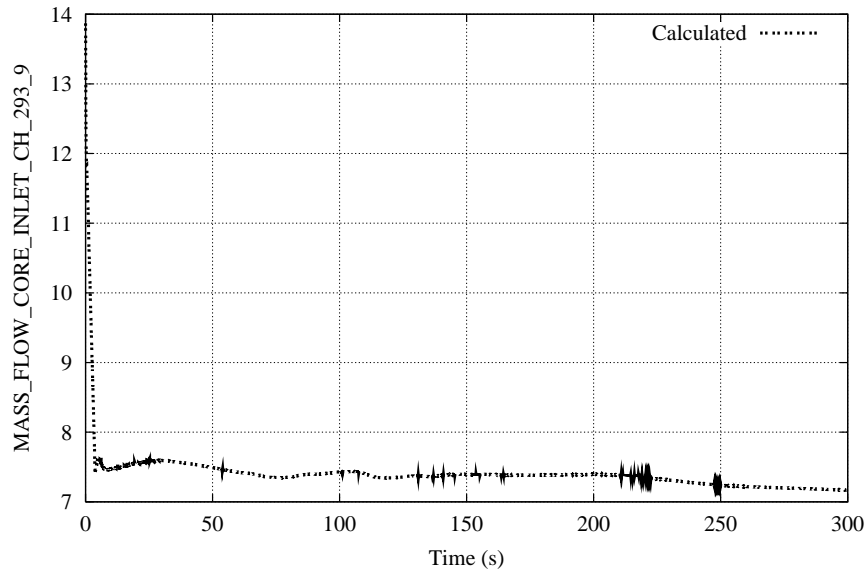
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



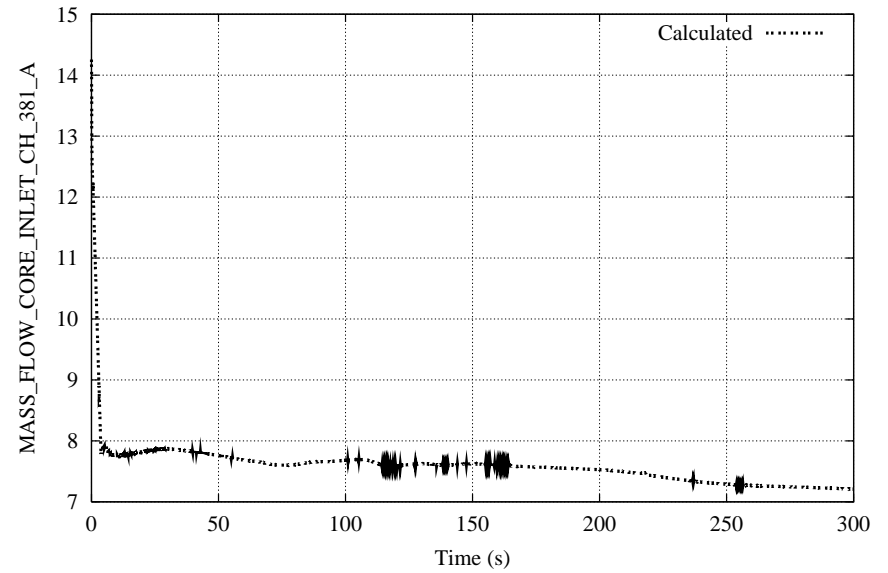
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



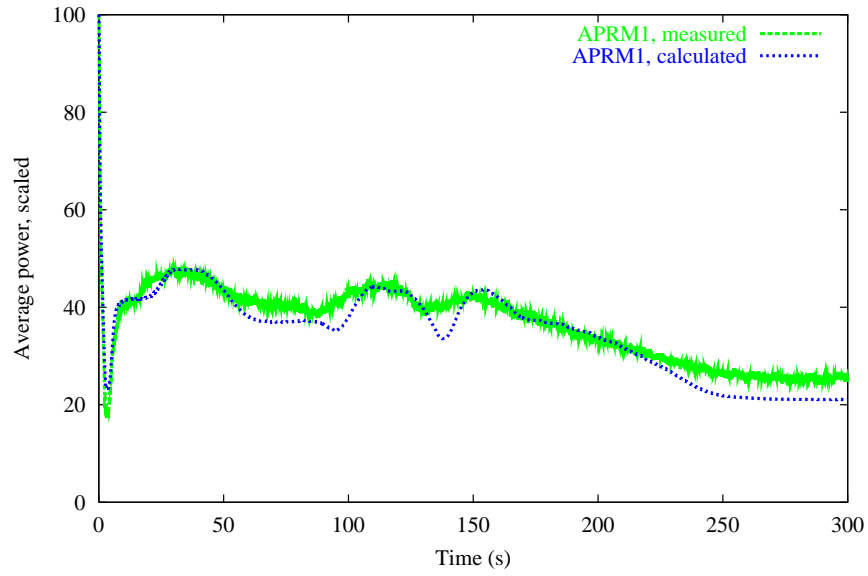
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



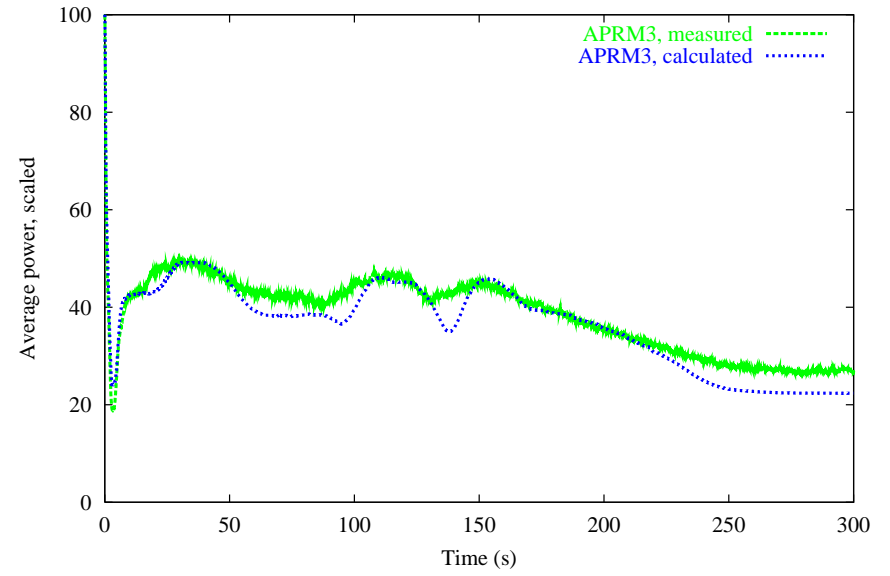
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



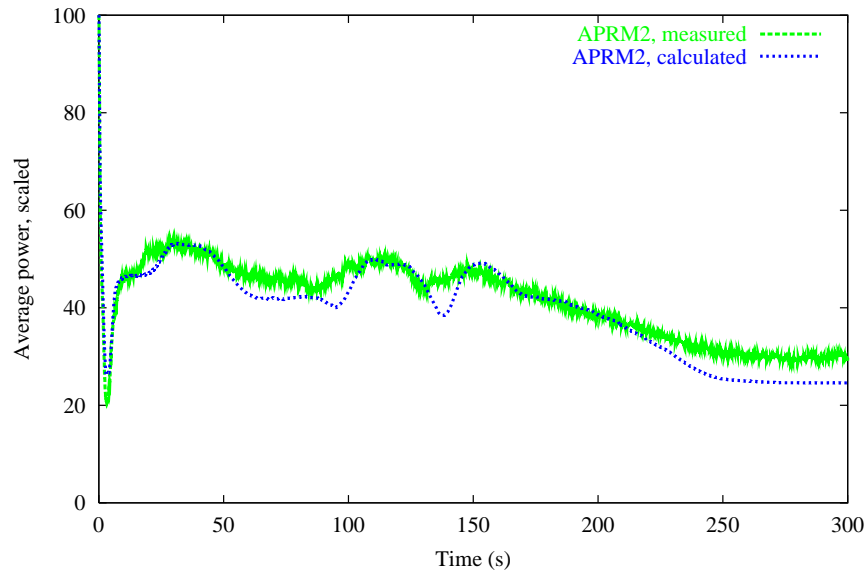
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



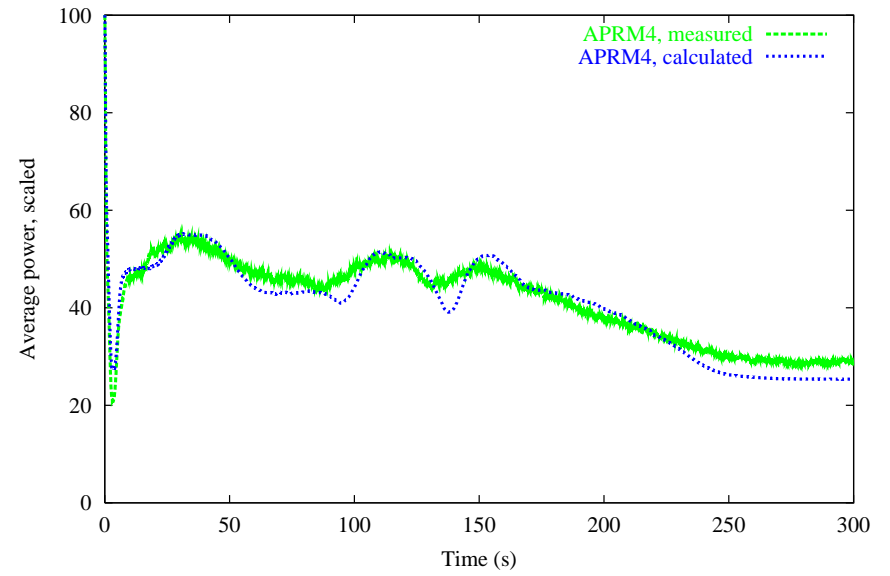
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



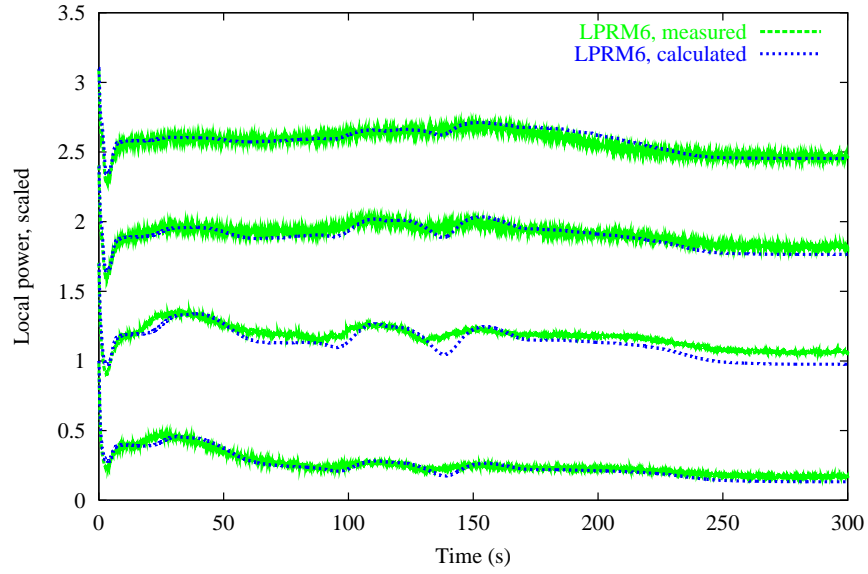
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



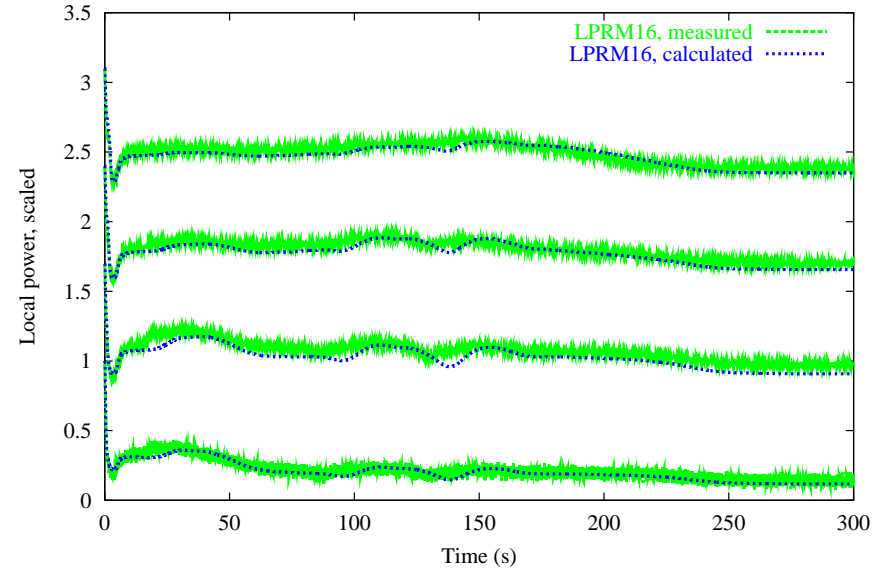
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



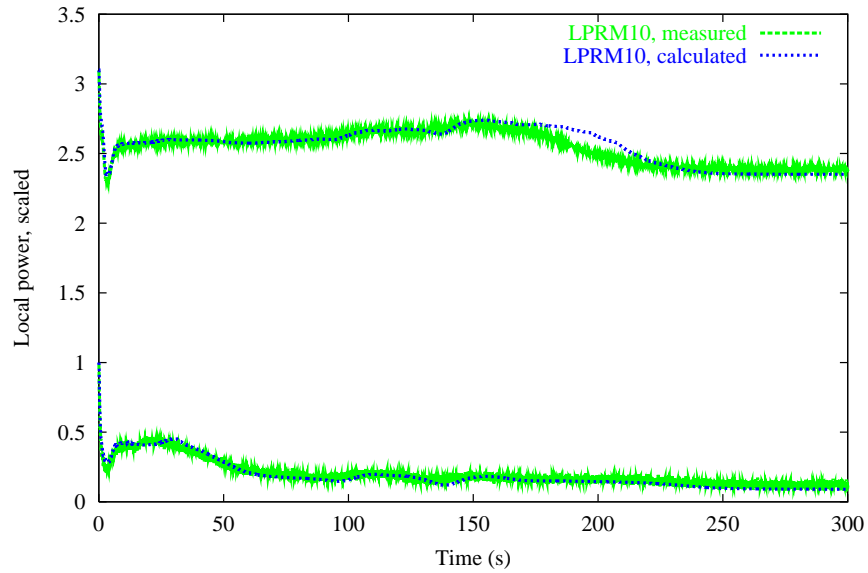
Olkiluoto 1 Load Rejection Test using 3s optimized valve boundary conditions



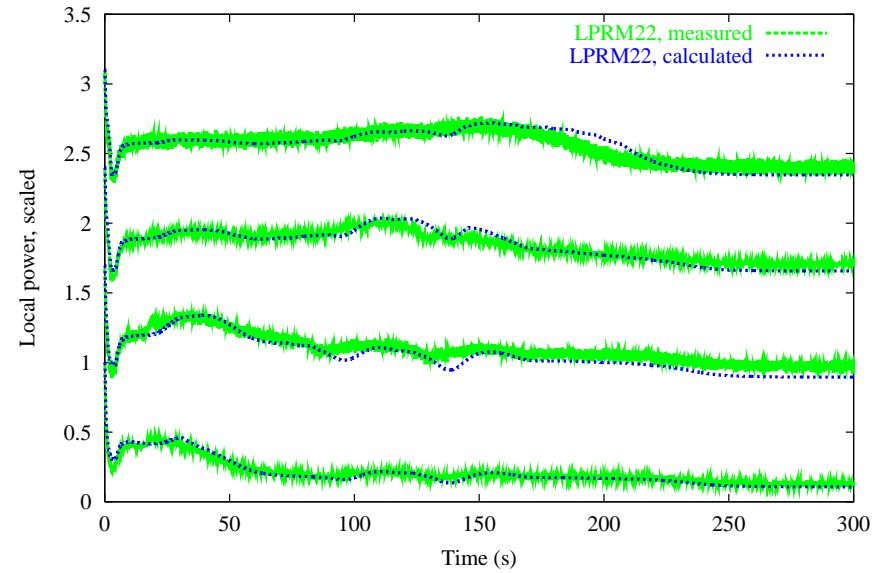
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